

# Meshless computational strategy for higher order strain gradient plate models

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Abstract: The present research focuses on the use of a meshless method for the solution of nanoplates by considering strain gradient thin plate theory. Unlike the most common finite element method, meshless methods do not rely on a domain decomposition. In the present approach approximating polynomials at collocation nodes are obtained by using radial basis functions which depend on shape parameters. The selection of such parameters can strongly influences the accuracy of the numerical technique. Therefore the authors are presenting some numerical benchmarks which involve the solution of nanoplates by employing an optimization approach for the evaluation of the undetermined shape parameters. Stability is discussed as well as numerical reliability against solutions taken for the existing literature.

Keywords: meshless; strain gradient; nanoplates; radial basis functions

### 1 Introduction

In the broad context of meshless methods Meshless Local Petrov-Galerkin (MLPG) [1] demonstrated to have strong capabilities to solve problems where weakly-singular traction and displacement boundary integral equations are involved. Moreover, other problems in this context have been analyzed and solved [2, 3]. The main idea of meshless methods is to go beyond the current limitations of finite element models for analyzing problems in mechanics [4]. Unlike classical FEM shape functions are developed for a scattered set of collocation nodes [4, 5] and their generation defines different meshless approaches. The one considered in the present work is named Radial Point Interpolation Method (RPIM) [6, 7, 8, 9] which has the fundamental property of Kronecker delta function [10, 11, 12, 13]. This approach makes the RPIM extremely easy to be implemented and used for the solution of any problem in structural mechanics.

It has been demonstrated to be extremely relevant in industrial applications for nanoengineering that nanoelectromechanical (NEMS) systems have been widely considered in the recent literature with several gradient elasticity problems [14, 15]. It has been demonstrated in recent literature that nano-structures are used for micro-sized systems and devices such as biosensors, nano-actuators and nano-electro-mechanical systems [16].

Among the vast literature of nonlocal theories stress-driven models have been recently presented for nanobeams presenting analytical solutions. For instance closed-form solution of Bernoulli and Timoshenko type for Erigen-like formulations and stress field and a bi-exponential averaging kernel functions characterized by a scale parameter is presented in [17, 18, 19]. Nonlocal strain gradient Bernoulli like beam models by



considering special bi-exponential averaging kernels and functionally graded materials has been presented in [20]. Nonlocal beam formulations have been presented within a thermodynamic framework, variational formulation within its analytical solution has been provided in [21]. Nanobeams can be subjected to axial loads which leads to buckling, such effects must be considered for a proper nanoengineering design, thus stress-driven buckling of nanobeams can be found in [22, 23]. In the context of dynamic problems vibrations in nonlocal integral elasticity has been recently considered for beams and plates in [24, 25].

Most of the nonlocal theories relies on homogenization approaches which aim at simplifying the problem by considering less modelling parameters in composite materials [26, 27]. It is remarked that size effects and microstructures paved the way in presenting innovative and multiscale approaches in solid mechanics [28, 29].

It has been demonstrated by several researchers that higher-order elasticity is becoming of paramount importance for solid mechanics as mentioned in [30, 31, 32]. In most researchers the term nonlocality has been brought by Eringen [33] and by Eringen and Edelen [34] where it can be found that the constitutive relations have to be modified to take into account their dependency on the mechanical properties of the entire body and not only of the properties in the neighbourhood of the material point. In this regard, in the present work nonlocal effects have be meaning introduced by by Altan and Aifantis [35, 36], which considered a simplified nonlocal model where all nonlocalities are concentrated in a gradient model similar to the one proposed by Mindlin [37]. Such approach, known as strain gradient theory, has been utilized also by others in other contexts in solid mechanics [38, 39, 40]. Strain gradient theory [41] has been demonstrated to be a constrained version of other higher-order unconstrained versions available in the literature such as couple stress [42, 43, 44] as well as micropolar theories [45, 46, 47].

In the framework of plate theories thin plate model is very common within the area of structural mechanics of investigating thin-walled structures [48, 49]. Such approach can be easily extended in order to consider composite structures such as laminated [50, 51] or sandwich ones [52].

In the present work the advantages of meshless methods are considered in the framework of strain gradient composite thin plate theory to solve such numerical problem.

### 2 Theoretical background

The present work considers the problem of laminated thin plates of rectangular plane form of size  $a \times b$ , where a, b indicate lengths with respect to x, y, respectively. The plate thickness is indicated with h and the plate represents the middle plane of the actual plate with h axis pointed normal to the h plate. Each ply which constitutes the stacking sequence is indicated with h and the total thickness is computed as  $h = \sum_{k=1}^{N_L} h_k$ , where h denotes the number of plies [53].

The present displacement field follows the classical thin plate theory and Cartesian displacements can be represented as [54]

$$\mathbf{U} = \mathbf{u} - z \mathbb{D}^{(s)} \mathbf{u},\tag{1}$$

where **u** is the vector collecting the three middle plane displacements of the material point and  $\mathbb{D}^{(s)}$  is the derivative operation defined in [55]. The strain vector  $\varepsilon$  is given by

$$\varepsilon = \varepsilon^{(m)} + z\varepsilon^{(b)},\tag{2}$$

in which  $\varepsilon^{(m)}$  and  $\varepsilon^{(b)}$  denote respectively the membrane and bending strains, respectively. They can be evaluated as follows

$$\boldsymbol{\varepsilon}^{(m)} = \mathbb{D}^{(m)} \mathbf{u}, \quad \boldsymbol{\varepsilon}^{(b)} = \mathbb{D}^{(b)} \mathbf{u}$$
 (3)

where the meaning of the differential operators  $\mathbb{D}^{(m)}$ ,  $\mathbb{D}^{(b)}$  is reported in [55] and [54].

BCs	x = 0, a	y = 0, b
Supported	$v = w = \frac{\partial w}{\partial v} = 0$	$u = w = \frac{\partial w}{\partial x} = 0$
Clamped	$u = v = w = \frac{\partial w}{\partial x} = \frac{\partial w}{\partial y} = 0$	$u = v = w = \frac{\partial w}{\partial x} = \frac{\partial w}{\partial y} = 0$
Free	No variables involved	No variables involved

**Table 1:** Essential boundary conditions considered.

Linear constitutive law is considered within the strain gradient theory as suggested in [40] allows to relate the membrane stresses in the k-th layer  $\sigma^{(k)}$  to the corresponding strain components  $\varepsilon$  as shown below [56, 57]

$$\boldsymbol{\sigma}^{(k)} = (1 - \ell^2 \nabla^2) \, \bar{\mathbf{Q}}^{(k)} \boldsymbol{\varepsilon},\tag{4}$$

in which the nonlocal parameter  $\ell$  includes the micro/macro-scale interaction effects. The dependency of the stresses on the strain distribution within the medium is emphasized by the presence of the Laplacian in Cartesian coordinate system:  $\nabla^2 = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2}$ . On the other hand,  $\bar{\mathbf{Q}}^{(k)}$  represents the plane stress-reduced stiffness coefficients matrix of the k-th layer. The terms  $\bar{Q}_{ij}^{(k)}$  of this matrix depend on the orthotropic properties of the layer (Young's moduli  $E_1$ ,  $E_2$ , Poisson's ratio  $v_{12}$  and shear modulus  $G_{12}$ ), as well as by an arbitrary orientation  $\theta^{(k)}$  as indicated in the book [58]. the  $\mathbf{S}_N$  and  $\mathbf{S}_M$  are the stress resultants that can be defined as follows

$$\mathbf{S}_{N} = \left(\mathbf{A}\mathbb{D}^{(m)} + \mathbf{B}\mathbb{D}^{(b)} - \ell^{2} \left(\mathbf{A}\mathbb{D}_{xx}^{(m)} + \mathbf{A}\mathbb{D}_{yy}^{(m)} + \mathbf{B}\mathbb{D}_{xx}^{(b)} + \mathbf{B}\mathbb{D}_{yy}^{(b)}\right)\right)\mathbf{u},$$

$$\mathbf{S}_{M} = \left(\mathbf{B}\mathbb{D}^{(m)} + \mathbf{D}\mathbb{D}^{(b)} - \ell^{2} \left(\mathbf{B}\mathbb{D}_{xx}^{(m)} + \mathbf{B}\mathbb{D}_{yy}^{(m)} + \mathbf{D}\mathbb{D}_{xx}^{(b)} + \mathbf{D}\mathbb{D}_{yy}^{(b)}\right)\right)\mathbf{u},$$
(5)

The differential operators  $\mathbb{D}_{xx}^{(m)}$ ,  $\mathbb{D}_{yy}^{(m)}$ ,  $\mathbb{D}_{xx}^{(b)}$ ,  $\mathbb{D}_{yy}^{(b)}$  are defined in [54]. The constitutive operators **A**, **B**, **D** represent instead the membrane, membrane-bending coupling and bending stiffness matrices of the laminated composite plates [58].

In the current paper, isotropic and composite schemes are considered, therefore for some configurations  $\mathbf{B} \neq \mathbf{0}$ , thus, membrane and bending behaviors are coupled. The variational form of the present equilibrium is represented by [54, 59]

$$0 = \int_{\Omega} \left\{ \left( \mathbb{D}^{(m)} \delta \mathbf{u} \right)^{T} \left( \mathbf{A} \mathbb{D}^{(m)} + \mathbf{B} \mathbb{D}^{(b)} \right) \mathbf{u} + \left( \mathbb{D}^{(b)} \delta \mathbf{u} \right)^{T} \left( \mathbf{B} \mathbb{D}^{(m)} + \mathbf{D} \mathbb{D}^{(b)} \right) \mathbf{u} \right.$$

$$\left. + \ell^{2} \left[ \left( \mathbb{D}_{x}^{(m)} \delta \mathbf{u} \right)^{T} \left( \mathbf{A} \mathbb{D}_{x}^{(m)} + \mathbf{B} \mathbb{D}_{x}^{(b)} \right) \mathbf{u} + \left( \mathbb{D}_{y}^{(m)} \delta \mathbf{u} \right)^{T} \left( \mathbf{A} \mathbb{D}_{y}^{(m)} + \mathbf{B} \mathbb{D}_{y}^{(b)} \right) \mathbf{u} \right.$$

$$\left. + \left( \mathbb{D}_{x}^{(b)} \delta \mathbf{u} \right)^{T} \left( \mathbf{B} \mathbb{D}_{x}^{(m)} + \mathbf{D} \mathbb{D}_{x}^{(b)} \right) \mathbf{u} + \left( \mathbb{D}_{y}^{(b)} \delta \mathbf{u} \right)^{T} \left( \mathbf{B} \mathbb{D}_{y}^{(m)} + \mathbf{D} \mathbb{D}_{y}^{(b)} \right) \mathbf{u} \right] - \delta \mathbf{u}^{T} \mathbf{q} \, d\Omega,$$

$$(6)$$

The present meshless technique is applied to such variational statement of the problem.

### 3 Radial point interpolation method (RPIM)

In the present approach solution is obtained in scattered points located in the given plate rectangular domain. The approximating polynomials involved possess the Kronecker delta property which allows a straightforward implementation of boundary conditions. In the following implementation, in plane displacements u and v, transverse displacement w as well as rotations  $w_x$ ,  $w_y$  are included in the numerical implementation even though more primary variables are involved in the present formulation [55, 54]. In the context of the strain gradient theory, the essential boundary conditions for the plate are shown in Table 1 where the four edges are identified by the values of the physical coordinates x and y.

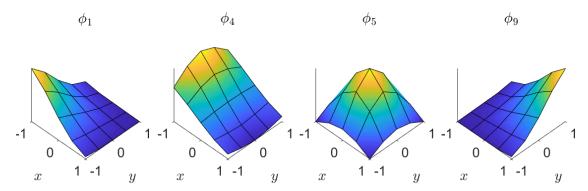
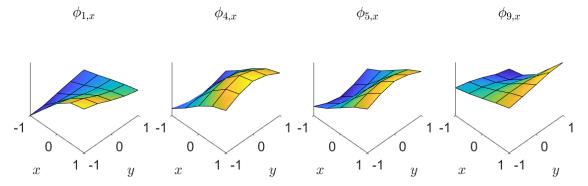


Figure 1: Sample of shape functions generation for a reference unitary domain.



**Figure 2:** Shape functions of Figure 1 derived with respect to x.

Any other higher-order derivative is carried out by numerical derivation of the aforementioned parameters. Since both the deflection and its first derivatives are considered unknown, the Hermite-RPIM formulation is here presented. Let's consider a domain enclosing n arbitrarily scattered nodes. The approximation of the generic displacement w(x,y) can be expressed as:

$$w(x,y) = \mathbf{R}^{\top}(\mathbf{x})\mathbf{a} + \mathbf{R}_{x}^{\top}(\mathbf{x})\mathbf{a}^{x} + \mathbf{R}_{y}^{\top}(\mathbf{x})\mathbf{a}^{y}$$
(7)

where  $\mathbf{R}$ ,  $\mathbf{R}_{,x}$  and  $\mathbf{R}_{,y}$  are the vectors including the radial basis functions (RBF) and their derivatives. The correspondent coefficients are indicated using vectors  $\mathbf{a}$ ,  $\mathbf{a}^x$  and  $\mathbf{a}^y$ . For the following numerical applications the well-known multi-quadrics (MQ) RBF is used in its general form

$$R_i(x,y) = [(x-x_i)^2 + (y-y_i)^2 + C^2]^q$$
(8)

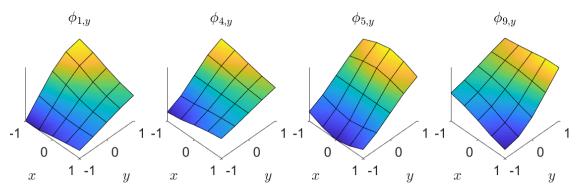
where  $C = \alpha_C d_c$ . Both q and  $\alpha_C$  are shape parameters that have to be tuned while  $d_c$  is the average nodal spacing.

The vectors of coefficients in Equation (7) can be obtained by enforcing the field function and its derivatives to be satisfied at all the n nodes falling within the support domain of the point of interest (x,y). The support domain is a local domain, typically circular or rectangular, centered in a point of interest which can either be a node or an integration point. This leads to 3n linear equations

$$\mathbf{W} = \begin{bmatrix} \mathbf{R} & \mathbf{R}_{,x} & \mathbf{R}_{,y} \\ \mathbf{R}_{,x} & \mathbf{R}_{,xx} & \mathbf{R}_{,xy} \\ \mathbf{R}_{,y} & \mathbf{R}_{,xy} & \mathbf{R}_{,yy} \end{bmatrix} \begin{pmatrix} \mathbf{a} \\ \mathbf{a}^{x} \\ \mathbf{a}^{y} \end{pmatrix} = \mathbf{G} \begin{pmatrix} \mathbf{a} \\ \mathbf{a}^{x} \\ \mathbf{a}^{y} \end{pmatrix}$$
(9)

where  $\mathbf{w}$ ,  $\mathbf{w}_{x}$  and  $\mathbf{w}_{y}$  are vectors of function values of the degrees of freedom considered in the collocation nodes. Thus, the independent parameter can be carried out as

$$w(x,y) = \left\{ \mathbf{R}^{\top} \quad \mathbf{R}_{,x}^{\top} \quad \mathbf{R}_{,y}^{\top} \right\} \mathbf{G}^{-1} \mathbf{W} = \mathbf{\Phi}^{\top} \mathbf{W}$$
(10)



**Figure 3:** Shape functions of Figure 1 derived with respect to y.

An example of what the shape functions look like as computed with this method is given in Figure 1. A squared domain, represented by  $3 \times 3$  regularly distributed nodes, is considered and all of the nodes are used to construct the shape functions for this domain. Figure 1 represents the shape functions and their first derivatives with respect to x and y for the nodes in the bottom left corner, in the middle left side, in the middle of the domain and in the top right corner respectively. For this particular case, the dimensionless parameters of the RBF are chosen as C = 1 and q = 0.05.

By following the same computational strategy of conventional finite element method [54] the algebraic form of the variational statement can be carried out because shape functions are evaluated at the collocation nodes. The needed integration is performed by following well-known Gauss integration rules. Solution of the present static problem is provided by Gauss elimination algorithm. The following section is dedicated to numerical results, stability and accuracy.

## 4 Applications

#### 4.1 Isotropic plates

This section shows the results of the numerical analysis of isotropic Kirchhoff nanoplates modelled according to the second-order strain gradient theory and analysed by means of a mesh free RPIM. In this case, there is no coupling between the in plane and out of plane behaviour. Hence, the only unknown variables considered in the numerical implementation the are transverse displacement w and the rotations  $w_x$ ,  $w_y$  The numerical codes are developed in MATLAB.

The results in terms of mid transverse displacement are presented in the nondimensional form as follows:

$$\bar{w} = \frac{1000wD}{q_z a^4} \tag{11}$$

where w is the central plate deflection,  $q_z$  is the magnitude of the transverse external load and D is the bending rigidity  $D = Eh^3/12(1-v^2)$ .

Nanoplates with different constraints are analysed, all having thickness h = 0.34 nm. Young's modulus and Poisson's ratio are taken as 1100 GPa and 0.3 respectively. Different nodal densities are also taken into account. Nanoplates represented by  $3 \times 3$ ,  $5 \times 5$ ,  $7 \times 7$  and  $11 \times 11$  equally spaced node grids are studied to analyse the convergence of the method. The local parameter  $\ell$  also varies according to the analysis and results presented in the available literature [60].

The support domain used for the mesh free implementation has rectangular shape and is centered in the Gauss points. Its dimensions are considered in the classical way [4]  $d_s = \alpha_s d_c$  where  $d_c$  is the average nodal spacing  $d_c = \sqrt{\Delta x^2 + \Delta y^2}$  and  $\alpha_s$  is a dimensionless parameter which, in this work, varies from 1.8 to 2.4.

BC	$\ell$ $(nm)$	Exact	Result	Error (%)
SSSS	0	4.0624	2.8622	29.5441
	0.2	4.0330	2.8338	29.7347
	0.5	3.8844	2.6956	30.6045
	1	3.4231	2.3142	32.3946
	0	1.2653	1.0716	15.3086
CCCC	0.2	1.2333	1.0555	14.4166
ccc	0.5	1.0979	0.9785	10.8753
	1	0.7946	0.7762	2.3156
	0	1.9171	1.5481	19.2478
SCSC	0.2	1.8783	1.5269	18.7084
SCSC	0.5	1.7093	1.4247	16.6501
	1	1.3040	1.1519	11.6641
	0	15.0113	13.5089	10.0085
CECE	0.2	14.9470	13.4511	10.0080
SFSF	0.5	14.6165	13.1711	9.8888
	1	13.5451	12.3957	8.4857
SCSF	0	11.2359	10.8262	3.6463
	0.2	11.1703	10.7635	3.6418
	0.5	10.8454	10.4568	3.5831
	1	9.8416	9.5731	2.7282

**Table 2:** Values of  $\bar{w}$  for  $3 \times 3$  nodal distribution, obtained for  $\alpha_C = 3$ , q = 1.3,  $\alpha_s = 2$ .

Note that, in this work,  $2 \times 2$  Gauss integration points are used in each cell of the background integration mesh.

Results listed in Tables 2-5 compare the available analytical solutions [60] with the present ones in terms of percentage error:

$$err_{\%} = 100 \frac{|w_e - \bar{w}|}{w_e}$$
 (12)

where  $w_e$  is the exact solution taken from the aforementioned references.

The provided comparison is performed for different boundary conditions, number of collocation nodes and nonlocal parameter values. The C and q coefficients characterising the MQ radial basis functions, as well as the nondimensional  $\alpha_s$  support domain parameter, vary as the number of nodes changes.

In addition, a visualization of the deformed configuration of the nanoplates here analysed is shown in Figure 9.

### 4.2 Composite plates

A similar analysis is also performed on squared cross-ply laminates. Simply-supported (SSSS) laminates with lamination schemes 0, (0/90),  $(0/90)_2$  and  $(0/90)_4$ , subjected to a sinusoidal load are analysed. The material property used are taken as  $E_1/E_2 = 25$ ,  $G_{12} = G_{13} = 0.5E_2$ ,  $G_{23} = 0.2E_2$ . Moreover, the thickness is given by h = a/100. As in the previous section, the results are compared in terms of nondimensional mid-deflection:

$$\bar{w} = w_0 \frac{E_2 h^3}{q_s a^4} \tag{13}$$

**Table 3:** Values of  $\bar{w}$  for  $5 \times 5$  nodal distribution, obtained for  $\alpha_C = 2.38$ , q = 0.01,  $\alpha_s = 2.4$ .

BC	$\ell$ $(nm)$	Exact	Result	Error (%)
SSSS	0	4.0624	3.8549	5.1078
	0.2	4.0330	3.8262	5.1277
	0.5	3.8844	3.6844	5.1488
	1	3.4231	3.2736	4.3674
	0	1.2653	1.3058	3.2008
CCCC	0.2	1.2333	1.2774	3.5758
cccc	0.5	1.0979	1.1466	4.4357
	1	0.7946	0.8384	5.5122
	0	1.9171	1.8851	1.6692
SCSC	0.2	1.8783	1.8509	1.4588
SCSC	0.5	1.7093	1.6903	1.1116
	1	1.3040	1.2914	0.9663
	0	15.0113	14.8721	0.9273
CECE	0.2	14.9470	14.8087	0.9253
SFSF	0.5	14.6165	14.5144	0.6985
	1	13.5451	13.7290	1.3577
SCSF	0	11.2359	11.2265	0.0837
	0.2	11.1703	11.1623	0.0716
	0.5	10.8454	10.8529	0.0692
	1	9.8416	9.9527	1.1289

where  $q_s$  is the magnitude of the sinusoidal load, taken as 1. The analysis is performed using  $11 \times 11$  regularly distributed nodes to represent the domain and  $2 \times 2$  Gauss points to perform the numerical integration. The results of the analysis, compared in terms of percentage error as shown in Equation 12, are shown in Table 6.

### **5 Conclusions**

In this work, strain gradient nanoplates, both isotropic and laminated, have been analyzed by means of the Radial Point Interpolation Method. The aim was to apply a RPIM formulation to thin plates modelled via strain gradient theory. Isotropic second order strain gradient Kirchhoff nanoplates with various boundary conditions are first analyzed. Numerical convergence with the analytical results achieved in recent literature was studied. In a similar way, cross-ply composite plates subjected to a sinusoidal load are also analysed. In this case, 11 × 11 nodes are used to represent the domain and different lamination sequences are considered. The paper provides a detailed explanation of the RPIM method theoretical and numerical implementation as well as theoretical notions in both explicit and matrix form. This work proves the validity of the RPIM for problems with higher order of derivatives involved.

**Table 4:** Values of  $\bar{w}$  for  $7 \times 7$  nodal distribution, obtained for  $\alpha_C = 2.38$ , q = 0.01,  $\alpha_s = 2.4$ .

BC	$\ell$ $(nm)$	Exact	Result	Error (%)
SSSS	0	4.0624	3.9617	2.4788
	0.2	4.0330	3.9310	2.5291
	0.5	3.8844	3.7815	2.6491
	1	3.4231	3.3461	2.2494
	0	1.2653	1.2796	1.1302
CCCC	0.2	1.2333	1.2470	1.1108
ccc	0.5	1.0979	1.1177	1.8034
	1	0.7946	0.8305	4.5180
	0	1.9171	1.9072	0.5164
SCSC	0.2	1.8783	1.8668	0.6123
SCSC	0.5	1.7093	1.7035	0.3393
	1	1.3040	1.3220	1.3804
SFSF	0	15.0113	14.9049	0.7088
	0.2	14.9470	14.8228	0.8309
	0.5	14.6165	14.5063	0.7539
	1	13.5451	13.7327	1.3850
SCSF	0	11.2359	11.2019	0.3026
	0.2	11.1703	11.1255	0.4011
	0.5	10.8454	10.8058	0.3651
	1	9.8416	9.9248	0.8454

**Table 5:** Values of  $\bar{w}$  for  $11 \times 11$  nodal distribution, obtained for  $\alpha_C = 2$ , q = 1.4,  $\alpha_s = 2.3$ .

BC	$\ell$ $(nm)$	Exact	Result	Error (%)
SSSS	0	4.0624	4.0472	0.3742
	0.2	4.0330	4.0174	0.3868
	0.5	3.8844	3.8711	0.3424
	1	3.4231	3.4424	0.5638
	0	1.2653	1.2794	1.1144
CCCC	0.2	1.2333	1.2464	1.0622
ccc	0.5	1.0979	1.1084	0.9564
	1	0.7946	0.8018	0.9061
	0	1.9171	1.9272	0.5268
SCSC	0.2	1.8783	1.8872	0.4738
SCSC	0.5	1.7093	1.7163	0.4095
	1	1.3040	1.3111	0.5445
	0	15.0113	11.2507	0.0566
SFSF	0.2	14.9470	11.1760	0.0214
2121	0.5	14.6165	10.8467	0.0089
	1	13.5451	9.9095	1.3761
SCSF	0	11.2359	11.2265	0.1317
	0.2	11.1703	11.1623	0.0510
	0.5	10.8454	10.8529	0.0120
	1	9.8416	9.9527	0.6899

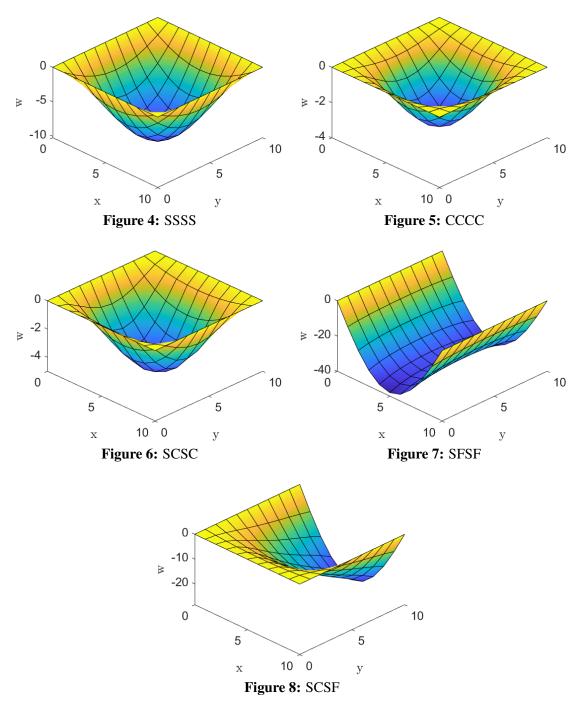


Figure 9: Deformed shapes of square isotropic nanoplates with different boundary conditions.

**Table 6:** Non dimensional values of  $\bar{w}$  mid deflection for composite nanoplates, obtained for  $\alpha_C = 1.85$ , q = -1.6,  $\alpha_s = 2.4$ .

$\ell$ $(nm)$	Laminate	Ref. [61]	Result	Error (%)
0	0	0.004312	0.004349	0.858071
	(0/90)	0.010636	0.010693	0.535916
U	$(0/90)_2$	0.005065	0.005109	0.868707
	$(0/90)_4$	0.004479	0.004520	0.915383
0.05	0	0.002170	0.002137	1.520737
	(0/90)	0.003931	0.004033	2.594760
0.03	$(0/90)_2$	0.002444	0.002351	3.805237
	$(0/90)_4$	0.002233	0.002167	2.955665
0.1	0	0.001450	0.001490	2.758621
	(0/90)	0.002522	0.002584	2.458366
	$(0/90)_2$	0.001623	0.001605	1.109057
	$(0/90)_4$	0.001490	0.001497	0.469799

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**Conflicts of Interest:** The authors declare that they have no conflicts of interest to report regarding the present study.

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