

## Supplementary materials for

# Mutual 3:1 Subharmonic Synchronization in a Micromachined Silicon Disk Resonator

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The equations of motion of two resonance modes with non-dissipative, cubic nonlinear bidirectional coupling are:

$$\begin{aligned}\ddot{x}_1 + \Delta\omega_1\dot{x}_1 + \omega_1^2 x_1 &= k_1 x_1^3 + k_2 x_1^2 x_2 + k_3 x_1 x_2^2 + k_4 x_2^3 + \lambda_1 \\ \ddot{x}_2 + \Delta\omega_2\dot{x}_2 + \omega_2^2 x_2 &= k_5 x_1^3 + k_6 x_1^2 x_2 + k_7 x_1 x_2^2 + k_8 x_2^3 + \lambda_2\end{aligned}\tag{S.1}$$

In these equations,  $x_1$  and  $x_2$  are the amplitudes of the first and second resonance modes with resonant frequencies of  $\omega_1 = 2\pi f_1$  and  $\omega_2 = 2\pi f_2$  respectively.  $\Delta\omega_1 = \omega_1 / Q_1$  and  $\Delta\omega_2 = \omega_2 / Q_2$  are the full width at half maximum (FWHM) bandwidths of the first and second resonance modes, respectively.  $\lambda_1$  and  $\lambda_2$  are external forces applied to the modes. The nonlinear stiffness terms  $k_1$  and  $k_8$  are cubic terms that result in Duffing nonlinearity in single DOF resonators, while  $k_2$  through  $k_7$  represent the cubic nonlinear coupling between the two modes. If  $k_2$  through  $k_7$  are zero, the equations represent two decoupled Duffing oscillators. As we are dealing with two oscillators that have a 3:1 frequency relationship, only the cubic nonlinear terms are included in the equations as these terms are the only ones that create third harmonics in the first-order approximation used here. These equations can be solved by a first-order approximation using a perturbation and multiple time-scales method with  $x_1 = 1/2 X_1 e^{i(\omega_1 t + \theta_1)} + 1/2 X_1 e^{-i(\omega_1 t + \theta_1)}$ ,  $x_2 = 1/2 X_2 e^{i(\omega_2 t + \theta_2)} + 1/2 X_2 e^{-i(\omega_2 t + \theta_2)}$ ,  $\lambda_1 = 1/2 \Lambda_1 e^{i(\Omega_1 t + \tau_1)} + 1/2 \Lambda_1 e^{-i(\Omega_1 t + \tau_1)}$ , and  $\lambda_2 = 1/2 \Lambda_2 e^{i(\Omega_2 t + \tau_2)} + 1/2 \Lambda_2 e^{-i(\Omega_2 t + \tau_2)}$ , where  $X_1$  and  $X_2$  are the oscillation amplitudes,  $\Lambda_1$  and  $\Lambda_2$  are the forcing amplitudes,  $\Omega_1$  and  $\Omega_2$  are the forcing frequencies, and  $\theta_1$ ,  $\theta_2$ ,  $\tau_1$  and  $\tau_2$  represent the phase of each signal. The solution results in the following four amplitude and phase equations in the base harmonic of each mode:

$$\begin{aligned}
8\omega_1\left(\frac{\Delta\omega_1}{2}X_1 + \dot{X}_1\right) &= 4\Lambda_1 \sin(\gamma_1) + k_2X_2X_1^2 \sin(\gamma_3) \\
8\omega_1X_1\dot{\theta}_1 &= -2k_3X_1X_2^2 - 4\Lambda_1 \cos(\gamma_1) - 3k_1X_1^3 - k_2X_1^2X_2 \cos(\gamma_3) \\
8\omega_2\left(\frac{\Delta\omega_2}{2}X_2 + \dot{X}_2\right) &= 4\Lambda_2 \sin(\gamma_2) - k_5X_1^3 \sin(\gamma_3) \\
8\omega_2X_2\dot{\theta}_2 &= -2k_6X_2X_1^2 - 4\Lambda_2 \cos(\gamma_2) - 3k_8X_2^3 - k_5X_1^3 \cos(\gamma_3)
\end{aligned} \tag{S.2}$$

These equations are four ordinary nonlinear differential equations. The first two are the amplitude and phase equations for the first mode, and the third and fourth equations are the amplitude and phase equations for the second mode.  $k_2$ ,  $k_3$ ,  $k_5$ , and  $k_6$  are coupling terms that appear in the solution and result in energy exchange between the two modes.  $k_1$ ,  $k_8$  and are the terms that result in internal resonance and consequently synchronization. The variables defined in the letter are listed here:

$$\begin{aligned}
K_1 &= \frac{\Lambda_1}{2\omega_1}, \delta_1 = \frac{k_2}{8\omega_1}, \alpha_1 = \frac{3k_1}{8\omega_1}, \beta_1 = \frac{k_3}{4\omega_1} \\
K_2 &= \frac{\Lambda_2}{2\omega_2}, \delta_2 = \frac{k_5}{8\omega_2}, \alpha_2 = \frac{3k_8}{8\omega_2}, \beta_2 = \frac{k_6}{4\omega_2}
\end{aligned} \tag{S.3}$$

Also in these equations:

$$\begin{aligned}
\gamma_1 &= \sigma_1 t + \tau_1 - \theta_1 \\
\gamma_2 &= \sigma_2 t + \tau_2 - \theta_2 \\
\gamma_3 &= \sigma_3 t + \theta_2 - 3\theta_1
\end{aligned} \tag{S.4}$$

where  $\sigma_1$  and  $\sigma_2$  are the frequency detuning parameters of the forces applied to the first and second modes:  $\Omega_1 = \sigma_1 + \omega_1$  and  $\Omega_2 = \sigma_2 + \omega_2$ . By having  $\gamma_1$  and  $\gamma_2$  fixed at  $90^\circ$ , the phase of each force input is fixed to excite the corresponding mode at the frequency that maximizes the vibration amplitude; in our experiments, the PLL provides this phase control. The resulting values of  $\sigma_1$  and  $\sigma_2$  in the numerical solution depend on the A-f dependence of each mode due to the Duffing nonlinear stiffness terms,  $k_1$  and  $k_8$  (also known as the backbone curve of the Duffing oscillator). By modeling the PLL in this manner, nonlinear damping terms (e.g. van der Pol self-sustained oscillators) are avoided, resulting in a more realistic oscillator model, and the PLL dynamics are omitted, making the model depend only on the micromechanical resonator's dynamics.  $\sigma_3$  is the frequency detuning parameter between the two resonance modes:  $\omega_2 = \sigma_3 + 3\omega_1$ . This detuning parameter is effectively the same parameter that we adjust through the

tuning voltage in our experiments. When synchronized, the phase difference between the two resonances ( $\gamma_3$ ) reaches a stable, constant value.

Among all the terms in Eq. S.2 and S.3,  $\alpha_1 X_1^2$ ,  $\beta_1 X_2^2$ ,  $\alpha_2 X_2^2$ , and  $\beta_2 X_1^2$  are easily measurable. By tracking the frequencies  $f_1$  and  $f_2$  as the amplitude of  $X_I$  is increased, one can measure  $\alpha_1 X_1^2$  and  $\beta_2 X_1^2$ , where  $\alpha_1 X_1^2$  is the change in frequency of  $f_1$  (known as Duffing A- $f$  dependence) and  $\beta_2 X_1^2$  is the change in frequency of  $f_2$  (known as coupling A- $f$  dependence). Similarly, by changing the amplitude of  $f_2$  and tracking the frequency of  $f_1$  and  $f_2$ , one can measure  $\beta_1 X_2^2$  and  $\alpha_2 X_2^2$ . Measured data for these parameters are presented in Fig. S.1.

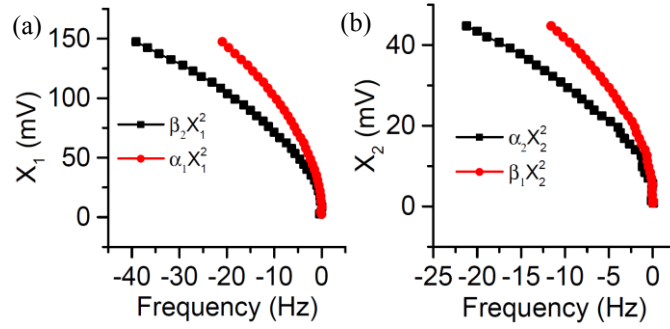


Fig. S.1: Measured A- $f$  dependence of the two modes due to Duffing nonlinearity and cubic nonlinear coupling.

Table S.1 lists the parameter values used in the simulations presented in Fig. 3 and Fig. 4 of the manuscript. The values of  $k_1$ ,  $k_3$ ,  $k_6$ , and  $k_8$  are estimated from the measured frequency response. The values of  $k_2$  and  $k_5$  are adjusted to fit the measured synchronization range. The values of  $k_1$ ,  $k_3$ ,  $k_6$ , and  $k_8$  do not change the synchronization range but they do shift the frequency of the maximum amplitude and affect the behavior of  $X_1$ ,  $X_2$ ,  $\theta_2 - 3\theta_1$ , and  $f_l$  (Fig. 4). Using the values listed in Table S.1, the simulated data presented in Fig. 3 and Fig. 4 can be reproduced.

Table S.1: Simulation parameters

<b>Parameter</b>	<b>Value</b>	<b>Units</b>
$\Delta\omega_1$	$2\pi\cdot 20$	rad/s
$\Delta\omega_2$	$2\pi\cdot 14$	rad/s
$k_1, k_8$	$5\cdot 10^{22}$	$1/(\text{m}\cdot\text{s})^2$
$k_2, k_5$	$1.39\cdot 10^{22}$	$1/(\text{m}\cdot\text{s})^2$
$k_3, k_6$	$3\cdot 10^{22}$	$1/(\text{m}\cdot\text{s})^2$
$\omega_1$	$2\pi\cdot 278000$	rad/s
$\omega_2$	$2\pi\cdot 834000$	rad/s
$\Lambda_1$	11.3 ... 21.2	$\text{m}/\text{s}^2$
$\Lambda_2$	1.5 ... 5.45	$\text{m}/\text{s}^2$
$\tau_1$	$-\pi/2$	rad
$\tau_2$	$-\pi/2$	rad

The small difference between the model and experiments at large drive amplitudes in Fig. 3 of the manuscript is attributed to the instability of PLL in synchronized region when the difference between  $\omega_1$  and  $\omega_2/3$  is high. Possible source of differences observed between the model and experiments in Fig. 4 is the uncertainties in the estimated parameters.