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Modelling the conditions for natural convection onset in open-cell porous Al/paraffin composite phase change materials: effects of temperature, paraffin type and metallic structure geometry

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1 **Modelling the conditions for natural convection onset in open-cell porous Al/paraffin**
2 **composite phase change materials: effects of temperature, paraffin type and metallic**
3 **structure geometry**

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10 **Abstract**

11 Composite Phase Change Materials (PCMs) can be made combining a PCM, i.e., a material
12 that is able store/release heat by its melting/solidification, and a low-amount of well distributed
13 high-melting and high-thermal conductivity phase with the aim of improving the overall
14 conductivity of the material and keeping its high heat storage capability. The composite made
15 by a paraffin and a porous structure of aluminium (Al) has been considered as the representative
16 of this material class. The design of these materials should not only take into account the
17 melting temperature (T_m) and the volume fraction of the paraffin, but also the geometrical
18 distribution and coarseness of the Al phase, which relate to the effective thermal conductivity
19 of the composite as well as the occurrence of natural convection once the PCM is in the molten
20 state. In the present paper, the inverse Body Centred Cubic (BCC) structure has been confirmed
21 to be the most suitable to model high porosity Al foams. For their BCC modelled structure, an
22 analytical equation is proposed for the evaluation of the overall thermal conductivity of the
23 composite PCMs. Also, new best fit equations for predicting permeability of BCC structure are
24 proposed. Analytical description is also given for the Rayleigh-Darcy number obtained as a
25 product of material-dependent term (related to T_m and volume fraction of PCM) and the
26 geometry dependent term (related to volume fraction of PCM, permeability as well as to
27 material coarseness alternatively given in terms of pores per inch, pore size or unit cell length).
28 The model has been validated by means of literature available experimental data. The proposed
29 simplified model can further be adjusted to correlate the onset of natural convection through
30 the local temperature gradient for the composite PCMs.

31 **Keywords**

32 Composite PCM; Paraffin; Effective thermal conductivity; Permeability; Convection.

33 **1 Introduction**

34 The phase change materials (PCMs) have been employed for thermal energy storage and
35 management in various fields, such as construction, textile, biomedical, solar energy,
36 electronics, batteries [1,2]. The low thermal conductivity of molten PCMs is a typically limiting
37 factor for their use, since it makes slower the thermal response of overall thermal energy storage
38 (TES) systems where PCMs are applied [1,2]. This is for example the case of paraffin waxes,
39 widely applied as PCMs in thermal energy storage systems. Paraffins offer high latent heat
40 fusion with the possibility to select the actual paraffin type on the basis of their melting
41 temperature of interest for the TES system at least in a relatively wide range [3].

42 The drawback of low thermal conductivity for a PCM material can be avoided by substituting
43 homogeneous PCMs by composite PCMs, where the actual PCM is combined with minor
44 amount of a phase characterized by high thermal conductivity and higher melting temperature
45 [1,2,4]. The high-conductivity phase can be both dispersed in the form of particles [5,6] or a
46 continuous phase [7,8]. Among these latter, metallic porous foams have been reported to have
47 many advantages: (i) the high porosity that can be reached allows high PCM volume fractions
48 and thus high energy storage capacity; (ii) their high surface/volume ratio allows fast heat
49 transfer to the PCM phase; (iii) the metallic structures offer structural functions also when the
50 PCM is completely molten [1,4].

51 The porous metallic structures, such as for example the complex and disordered foams are
52 usually modelled as lattices, obtained by the regular repetition of unit cells such as the
53 tetrakaidekahedral [9], body-centred cubic (BCC) [9–12], the face-centered cubic [9,12] ones.
54 In the present study, the BCC structure is considered, as previously suggested to be the closest
55 to real foam [10,11].

56 In the stage of the composite PCM design, the possibility to have or avoid natural convection
57 of the molten PCMs has to be also taken into account. As a matter of fact, depending on specific
58 part needs and configuration of the TES system, natural convection of PCMs within the PCM
59 composite material can have both favourable and adverse effects. Natural convection in the
60 molten PCM enhances heat transfer during melting process [13], leading to faster responses of
61 the TES system in terms of energy storage/release, as proved experimentally and predicted
62 numerically [14,15]. On the other hand, in the case of possibility for natural convection to take
63 place within a molten medium, the overall thermal response can be effected by the heat source

64 location [16]. In the case of composite PCMs, this also corresponds to a dependence on the
65 spatial arrangement of composite phases [17,18]. Finally, natural convection phenomena cause
66 orientation-dependent heat transfer which could be unsuitable in some engineering components
67 [18]. Similarly, in the special class of PCM composite such as the nano-encapsulated PCMs
68 (NEPCMs) dispersed in a fluid forming nanofluids with PCM properties, Ghalambaz et al.
69 [19,20] observed the effect of orientation of heat flow as well as changes occurring in
70 charging/discharging and, thus, in the effectiveness the TES system induced by different
71 amounts of NEPCMs.

72 The effects of natural convection in metallic-foam composites PCM has been investigated
73 by several authors [21,22]. Specific studies focus on the presence/suppression [22,23] or on
74 the impact of natural convection of PCM in porous media or in more complex systems or
75 devices containing these porous system [24]. Zhang et al. [21] presented cases of composite
76 PCMs and service conditions where natural convection effect cannot be neglected although
77 the heat conduction is dominant. Lafdi et al. [25], who experimentally studied the phase
78 change heat transfer process of paraffin in aluminium foams characterized by porosity in
79 88.4-96.6% range, reported that effect of convection is greater in higher porosity and bigger
80 pore size foams. However, on the basis of experimental or simulated tests, some authors
81 stated that natural convection in molten PCMs has negligible effects. The considered cases
82 were Al foam-PCM with 93.5%-94.5% porosity and pore density of 10 pores per inch (PPI)
83 [26], Al foam-PCM with 80% and 95% porosity and pore density of 20 PPI [27], Cu foam-
84 PCM with the porosity of 90% and pore density of 30 PPI [16]. To the authors' knowledge,
85 no organic view is currently available for the combined effects of phases volume fraction
86 and type, porous coarseness, and orientation on the onset of natural convection in composite
87 PCMs. On the other hand, the effect of orientation, size and local geometry of the container
88 of PCM materials on natural convection have been into account by the research groups of
89 Chamkha [19,28–30].

90 Fluid dynamic studies related to porous media for different engineering applications (see as
91 example Soboleva [31], Poulikakos et al.[32]) show that conduction heat transfer domain as
92 well as the presence of natural convection are correlated to Rayleigh-Darcy (R_D) number.
93 Specifically, where a porous medium filled by liquid is heated from its bottom, natural
94 convection in the liquid phase occurs above a critical value that is close to 40 [31,33]. Where
95 heat is at a lateral surface, with constant temperature, critical R_D number decreases to 1 [32]

96 with a stronger tendency for natural convection to arise and contributing to the overall heat
97 exchange.

98 The Rayleigh-Darcy number is defined as follows:

$$99 \quad \mathbb{R}_D = \frac{g\Delta\rho KH}{\alpha_{eff}\mu} = \frac{g\rho\beta\Delta T KH}{\alpha_{eff}\mu} \quad (1)$$

100 where, H [m] is the specimen height, K [m²] is the permeability of the structure, μ [Pa*s] is the
101 dynamic viscosity of the fluid phase, $\Delta\rho$ [kg/m³] is the density difference of the fluid, α_{eff} [m²/s]
102 is the effective thermal diffusivity of the composite and g [m/s²] is gravitational acceleration.
103 In the case of a direct correlation between density, temperature changes, and temperature-
104 independent coefficient of volumetric thermal expansion β [1/K] (equal to three times the
105 coefficient of linear thermal expansion), the density difference can be expressed in terms of the
106 temperature difference ΔT [K]. In the case, constant β and the average density ρ are used, the
107 equation modifies as written above. Further, the effective diffusivity of the composite can be
108 substituted by the ratio between its effective thermal conductivity λ_{eff} and the thermal inertia
109 I_{eff} :

$$110 \quad \mathbb{R}_D = \left[\frac{g\beta\rho}{\mu} \frac{I_{eff}}{\lambda_{eff}} \right] \times (KH) \times \Delta T = M \times (KH) \times \Delta T \quad (2)$$

111 Eq. 2 can thus be split into three terms: the temperature difference ΔT , a material-dependent
112 term (M) and a purely geometrical term (K×H). By using Eq.2, designers can check if critical
113 \mathbb{R}_D values are exceeded or not on the basis of the geometrical features and properties of the
114 porous metallic structure as well as on the properties of the fluid. Nevertheless, to the authors'
115 knowledge, up to now no simple tool is available for designers in view of the selection of
116 suitable geometrical features for the porous metallic structure (such as coarseness and porosity)
117 that allow the presence/absence of natural convection in a composite PCM where the PCM
118 type has been already identified based on its melting temperature.

119 Actually, this new approach can be followed by considering that the \mathbb{R}_D index is the product of
120 three terms, as shown in Eq.2. The first multiplying factor (M) relates to the properties of the
121 fluid which natural convection motion can occur (i.e., the PCM in its molten phase) as well as
122 to the properties of the composite. The second factor (K*H) is related to the geometrical
123 features of the structural part of the composite PCMs.

124 The present paper is aimed at giving an overall view of the roles played by geometrical features
125 of the porous structure (such as structure coarseness, porosity, relative component thickness in

126 the direction of heat flow) on the presence of natural convection in this molten phase of a
 127 composite PCM containing a high-melting porous structure. The present paper is specifically
 128 focused on composite PCMs made of a paraffin as active phase characterized by melting
 129 temperature T_m and an aluminium porous foam as structural phase, modelled as an inverse
 130 BCC lattice. The aim is to give the possibility to calculate the R_D number by simple but accurate
 131 analytical models so that the designer can easily predict if the existence of natural convection
 132 in a composite PCM. Even if related to paraffin/aluminium composite PCMs, the paper
 133 supplies the method to derive the parameters of interest to estimate conditions leading to natural
 134 convection in other composite PCMs containing high-melting porous structures.

135 **2 Modelled geometry of the two-phase structure of composite PCMs**

136 The aluminium open cell porous structure is modelled as a lattice formed by regularly 3D
 137 stacked inverse body-centred cubic (BCC) unit cell. This means that aluminium stays in the
 138 volume not occupied by spheres in a BCC unit cell where spheres of diameter d are placed at
 139 the vertices and in the central points of a cube having side length L (Fig. 1a), also referred as
 140 lattice length. The open cell structure is continuous when:

$$141 \quad \frac{\sqrt{3}}{2}L < d < \frac{3}{2\sqrt{2}}L \quad (3)$$

142 The extreme cases for the open cell Al structure corresponding to the lowest and highest
 143 diameter are shown in Fig. 1b. The smallest diameter d corresponds to the case of tangent
 144 spheres and to the highest volume fraction of the open-cell structure. The highest diameter
 145 corresponds to minimal cross-sectional area and to the lowest volume fraction of the open-cell
 146 structure.

147 In the modelled composite PCMs, the volume occupied by interconnected spheres is filled by
 148 the PCM. Focusing on the PCM phase distribution, the model for PCM composite is often
 149 referred as the BCC model [9–12]. The porosity ε of the BCC cell thus corresponds to the
 150 volume fraction of the PCM in the composite material. The calculation of the porosity can be
 151 easily done when the overlapping of sphere volumes is taken into account. As shown in Fig.1c,
 152 there are two types of overlapping spheres, each couple consisting of two equal spherical caps
 153 whose single volume can be calculated as:

$$154 \quad V = \frac{\pi \times h^2}{3} \left(\frac{3}{2}d - h \right) \quad (4)$$

155 where, h is the height of spherical cap. Two types of spherical caps and two corresponding
 156 numbers have to be considered: h_1 refers to the case of overlapping between the central sphere
 157 and one centred at a cube vertex (16 in total), h_2 refers to the overlapping of two spheres centred
 158 on vertices or the caps falling outside the cube cell ($6*2$). When d is lower than L, there is no
 159 overlap between spheres located in vertices (thus $h_2=0$). As a result, the porosity (i.e., PCM
 160 volume fraction) ε can be computed by following equation:

$$161 \quad \varepsilon = \frac{\frac{1}{3}\pi d^3 - 16\pi h_1^2 \left(\frac{d}{2} - \frac{h_1}{3}\right) - 12\pi h_2^2 \left(\frac{d}{2} - \frac{h_2}{3}\right)}{L^3}$$

$$162 \quad = \frac{1}{3}\pi \left(\frac{d}{L}\right)^3 - 16\pi \left(\frac{h_1}{L}\right)^2 \left(\frac{d}{2L} - \frac{h_1}{3L}\right) - 12\pi \left(\frac{h_2}{L}\right)^2 \left(\frac{d}{2L} - \frac{h_2}{3L}\right) \quad (5)$$

163 where, $h_1 = (2d - \sqrt{3}L)/4$. $h_2 = (d-L)/2$ for $d > L$ (ε is about 0.9395) otherwise $h_2=0$. Since the
 164 length size L will be hereafter considered as the geometry parameter representative of the
 165 metallic structure coarseness, the second term of the above equation has been rearranged in
 166 terms of cell side length normalized values: d/L , h_1/L and h_2/L .

167 The range of porosity ε in the model adopted can be obtained by combining Eq.3 and Eq.5,
 168 corresponding to $0.6802 < \varepsilon < 0.9945$. Within this range, for each ε only one d/L solution of Eq.5
 169 is possible, with d constrained to be positive and the condition for open-cell foam (Eq.3). A set
 170 of d/L has been calculated in this way by the authors and is plotted in the Fig.2. It can be
 171 observed that d/L almost uniformly increases with porosity before ε is 0.9395, then increase
 172 faster. In order to simplify the calculation of d/L , analytical expressions for d/L with ε have
 173 been derived by best fitting the dataset of points calculated via Eq.5. The following $d/L=f(\varepsilon)$
 174 equations are obtained for h_2 equal to or different from zero, respectively:

$$175 \quad \frac{d}{L} = 0.4665\varepsilon^2 - 0.2452\varepsilon + 0.8179 \quad \text{for } 0.6802 < \varepsilon < 0.9395 \quad (6)$$

$$176 \quad \frac{d}{L} = 14.855\varepsilon^2 - 27.639\varepsilon + 13.857 \quad \text{for } 0.9395 < \varepsilon < 0.9945 \quad (7)$$

177 The coefficient of determination (R^2) is 0.9999 and 0.9902, respectively.

178 Alternatively, a further best fit equation is derived for the whole porosity range for open cell
 179 inverse BCC structure:

$$180 \quad \frac{d}{L} = 4.969\varepsilon^3 - 11.683\varepsilon^2 + 9.599\varepsilon - 1.825 \quad \text{for } 0.6802 < \varepsilon < 0.9945 \quad (8)$$

181 The coefficient of determination (R^2) is 0.998.

182 This latter correlation leads to a maximum relative error of around 0.6% at $\varepsilon=0.994$ and thus
183 can be used when rough estimations of d/L can be considered. All the $d/L=f(\varepsilon)$ correlations
184 above (Eq.6 to 8), plotted in Fig.2, are independent from the coarseness of the porous structure,
185 which can be linked both to L and d .

186 **3 Material properties and modelled material dependant term M**

187 Once the geometry of the phases has been defined, the temperature-dependence of
188 thermophysical properties of single phases or the effective properties of the composite PCM of
189 interest for the calculation of the M term in Rayleigh-Darcy number need to be considered.

190 *3.1 Material properties*

191 Following the results of a previous paper by the same authors [34], the density (ρ_{PCM}), specific
192 heat ($C_{p,PCM}$), thermal conductivity (λ_{PCM}) in their molten range for paraffins having the
193 melting temperature (T_m) in the range 290-380K can be easily analytically calculated.
194 Equations allow to calculate the properties on the basis of T_m , that is the parameter on the basis
195 of which a paraffin to be used as PCM is selected, as described in recently work by Li [34].

196 Among many paraffins available, $C_{18}H_{38}$ or $C_{28}H_{58}$ (melting temperatures 300K and 334K,
197 respectively) have been considered for comparison purposes. Their calculated temperature-
198 dependent properties are compared in Fig.3. It can be easily observed that the specific heat of
199 these molten paraffins can be considered as practically independent on paraffin type. Slight
200 dependence on paraffin chemistry can be noticed for density and thermal conductivity.

201 The property of main interest for the present paper related to the onset of natural convection is
202 the dynamic viscosity of molten paraffins (μ [Pa*s]). Li [34] suggested that the temperature-
203 dependence of dynamic viscosity can be derived from T_m on the basis of the following couple
204 of equations [34]:

$$205 \mu = 38.78 \times \exp(-18.83(T/T_b)) + 0.01426 \times \exp(-4.329(T/T_b)) \quad (9)$$

206 where, T_b represents the boiling temperature for the paraffinic PCM, related to its melting
207 temperature as follows:

$$208 T_b = 4.56 \times 10^{-5} \times T_m^3 - 0.02611 \times T_m^2 + 6.341 \times T_m - 201.366 \quad (10)$$

209 As shown in Fig.3b, for the two paraffins here considered, strong temperature- as well as
210 paraffin-type dependence can be observed.

211 In the temperature range of interest for considering the onset of natural convection in the
 212 modelled composite, which can be considered to range from the melting temperature of the
 213 paraffins to 100K in excess of them, the temperature-dependent thermophysical properties of
 214 Al are also given in [34]. For example, the values of properties for pure Al phase are calculated
 215 at the melting temperatures of two paraffins (300K and 330K): $\rho_{Met} = 2699.6 \text{ kg/m}^3$ and 2693.0
 216 kg/m^3 for $C_{18}H_{38}$ or $C_{28}H_{58}$ respectively, while at T_m of the same paraffins $C_{p,Met}$ is 891.0 and
 217 $907.8 \text{ J/(kg}\cdot\text{K)}$, λ_{Met} is 258.7 and 254.7 $[\text{W}/(\text{m}\cdot\text{K})]$.

218 3.2 Effective thermophysical properties

219 In order to calculate the material-dependent term M in Eq.2, the effective thermal inertia I_{eff}
 220 and the effective thermal conductivity λ_{eff} of the composite PCM have to be derived. The
 221 effective thermal inertia of a multiphase material containing n different phases can be easily
 222 calculated as:

$$223 \quad I_{eff} = (\rho C_p)_{eff} = \sum_{j=1}^m \rho_j C_{p_j} f_j \quad (11)$$

224 where ρ_j $[\text{kg}/\text{m}^3]$, C_{p_j} $[\text{J}/(\text{kg}\cdot\text{K})]$, f_j are density, specific heat and volume fraction of the j^{th} phase
 225 of the m phases [35]. This formulation could allow to consider the PCM phase transition for
 226 the composite PCM, where three phases are present. Nevertheless, since in the present paper a
 227 completely molten PCM with volume fraction ε is considered, I_{eff} can be simply calculated as:

$$228 \quad I_{eff} = \rho_{Met} \times C_{p,Met} (1 - \varepsilon) + \rho_{PCM} \times C_{p,PCM} \times \varepsilon \quad (12)$$

229 On the contrary, the calculation of the effective thermal conductivity of multiphase materials
 230 is not straightforward. As a matter of fact, λ_{eff} is not only a function of thermal conductivity
 231 and volume fractions of phases, but also of their geometric distribution. Thus, only rough
 232 estimation of λ_{eff} can be obtained by applying the mixture rules for phases distributed in parallel
 233 or series, which correspond to the maximum/minimum thermal conductivity values [36].
 234 Several analytical equations have been proposed during the years for dilute second phase
 235 inclusions [36] or for regularly arranged continuous phases [37–41].

236 Fig.4 shows the ε -normalized effective thermal conductivity (NETC) correlations in the whole
 237 porosity range for which continuous open-cell structure is possible for (inverse) BCC lattice of
 238 the high-conductivity phase (a) and for the high-porosity range (b). NETC obtained by direct
 239 simulation (DS) from authors' work [34] for the $C_{18}H_{38}/\text{Al}$ or $C_{28}H_{58}/\text{Al}$ composites at the
 240 melting temperatures of the paraffins are presented as datapoints (black and red star symbols).
 241 The results are close to those calculated in literature using the same approach for $C_{22}H_{46}/\text{Al}$,

242 represented by green star symbols [23]. This is due to very small difference in the thermal
243 conductivity of the paraffins, the ratios $\lambda_{low}/\lambda_{high}$ between the low and high-conductivity phase
244 at the temperatures considered is very close (of about 5.9×10^{-4} , value that should be modelled
245 when $\varepsilon=1$). Since the same ratio will be closer for different paraffins and temperatures
246 considered in the present paper (not exceeding 480K), the results from DS simulation will be
247 very close to those ones. The same holds good for DS calculated PCM composite formed by
248 combining a paraffin with Cu (for which a thermal conductivity close to 400 W/(m K) is
249 generally reported [43,44] and $\lambda_{low}/\lambda_{high}$ of about 3.8×10^{-4} is considered), as shown in purple
250 star symbols. In order to show the slight, but still existing, effect of $\lambda_{low}/\lambda_{high}$, plots in Fig.4 also
251 include the results from DS simulation for air/Al and water/Al composites where Al porous
252 structure is modelled as the BCC lattice [10]. In these cases, the $\lambda_{low}/\lambda_{high}$ are 1×10^{-4} and 28×10^{-4} ,
253 for air and water, respectively and results (blue and yellow star symbols, respectively) lie
254 across those for paraffin/metallic structures as reasonably expected. The combination of these
255 results shows that also for thermal conductivity ratios varying of 1 order of magnitude, the
256 effect of their change is smaller than that induced by limited changes in porosity.

257 The above data are then compared in the same figures with experimental data available in
258 literature for the same combinations of materials and metallic open-cell structures with high
259 porosity (open symbols, where in the plots the same colour is adopted for each combination of
260 phases. These results confirm the slight effect of $\lambda_{low}/\lambda_{high}$, but also show that BCC structures
261 well predict the NETC for composites with a high-conductivity open cell structure (about above
262 0.93) but overestimate NETC for lower porosity. In the porosity below 0.8, a foam becomes
263 more close-cell than open-cell structure [36].

264 Other lattice description could be adopted for the metallic open-cell structure of the high-
265 conductivity phase of a composite PCM, such as (inverse) face centred cubic (FCC) or simple
266 cubic (SC). Calculations for their effective thermal conductivity were performed by Rawson et
267 al. [42] adopting a lattice Monte Carlo (LMC) approach (leading to a direction-averaged λ_{eff}).
268 For the $\lambda_{low}/\lambda_{high}$ of the same phases coupled in the composite structure ($C_{18}H_{38}/Al$), the results
269 are plotted with black triangular symbols in Fig.4. Their match with experimental data is
270 suitable only for lower than 0.9, while for high- ε these models clearly underestimate NETC.
271 The better fit of BCC model for high- ε has been reported by some authors to be correlated to
272 the minimum surface-area to volume fraction of BCC corresponding to the physical
273 phenomenon of pore formation in foams [10,11].

274 Alternative lattice descriptions of the high-conductivity open-cell structure have been proposed
 275 in literature. Several authors considered structures made of constant-section bars, leaving free
 276 one or a few parameters then identified by fitting experimental results [37–41]. While the
 277 approach allows a good description of single cases, specifically at high porosity, their
 278 application to paraffin/Al porous foam is proved to result in widely scattered NETC for the
 279 high porosity values and in an unacceptably low NETC for $\varepsilon < 0.88$. These methods are thus not
 280 considered there, and their predictions are not shown in Fig.4.

281 As an overall result from the comparisons made in Fig. 4a and 4b, it can be observed that at
 282 relatively high porosity the BCC description is quite good, for the range of ratio between
 283 thermal conductivity used in the DS for paraffin/Al or paraffin/Cu are in a narrow range. While
 284 the predictions supplied by BCC model for open-pore structures produced by the conventional
 285 methods are not suitable as porosity reduces, BCC remains the lattice with the highest NETC
 286 for a given porosity and could thus be considered as the target one in some cases such as the
 287 composite PCMs considered in the present paper. Its selection as a reference model has thus
 288 been confirmed. The description further allows to consider structure with open cell porosity
 289 and relatively high NETC in a wide range of porosity (down to $\varepsilon = 0.69$). For this BCC lattice
 290 the following analytical description (modified by Hu after Progellhof model [23,46] can be
 291 adopted:

$$292 \quad \lambda_{eff} = \varepsilon \times \lambda_{PCM} + (1 - \varepsilon)^\theta \times \lambda_{Met} \quad (13)$$

293 where, θ is a fitting constant, identified as 1.3142 by Hu [23] for a $C_{22}H_{46}/Al$, and as 1.3296 by
 294 the authors of the present paper [34]. This latter fitting index has been identified by λ_{eff} results
 295 provided by direct simulation (DS) at 2 temperatures (T_m of paraffin and ($T_m + 100K$)) for
 296 composite PCMs obtained by combining $C_{18}H_{38}$ or $C_{28}H_{58}$ paraffins with the inverse BCC
 297 structures made of Al or Cu with porosity ranging from 0.69 to 0.98.

298 *3.3 Modelled material dependant term M*

299 The values of M have been calculated for the composite PCMs made by combining the Al
 300 porous structure with different porosity with paraffins $C_{18}H_{38}$ or $C_{28}H_{58}$ (melting temperatures
 301 300K and 334K, respectively). This Eq.13 has been used together with Eq. 12 for I_{eff} , Eq. 9
 302 and 10 for dynamic viscosity and the following one

$$303 \quad \Delta\rho_{PCM} = 209.5 \times (\Delta T)/T_m \quad (14)$$

304 derived from the description of molten paraffin density [34], where $\Delta\rho$ is proportional to the
 305 temperature difference. The overall equation for M calculation is the combination of the
 306 following and Eq (10) for T_b .

$$307 \quad M = \frac{F}{\mu} = \frac{\frac{209.5g}{T_m} \times \frac{(\rho Cp)_{Met} \cdot (1-\varepsilon) + (\rho Cp)_{paraffin} \cdot \varepsilon}{(\lambda_{Met} (1-\varepsilon)^{1.3296} + \varepsilon \lambda_{paraffin})}}{38.78 \times \exp(-18.83T/T_b) + 0.01426 \times \exp(-4.329T/T_b)} \quad (15)$$

308 Further, a new variable M' is considered, as the ratio between M at a given temperature and
 309 the value of M assumed at the melting temperature of the molten (PCM, paraffin) phase, so
 310 that M' can be described as:

$$311 \quad M' = M/M_{T_m} = (F \times \mu_{T_m}) / (F_{T_m} \times \mu_T) \quad (16)$$

312 Calculations are performed for temperatures ranging from the melting and boiling one for each
 313 paraffin, and the results are shown in Fig. 5a. The strong temperature dependence of M' , and
 314 thus of M and the effect of the molten composite are clear.

315 In order to investigate the impact of temperature-dependence of properties included in M, a
 316 reference case in which only the dynamic viscosity of paraffin is temperature-dependent, while
 317 other properties keep constant their values at the melting temperature of the PCM used is
 318 considered. In this case M' is equal to:

$$319 \quad M'_{T_m} = \mu_{T_m} / \mu_T = \frac{38.78 \times \exp(-18.83T_m/T_b) + 0.01426 \times \exp(-4.329T/T_b)}{38.78 \times \exp(-18.83T/T_b) + 0.01426 \times \exp(-4.329T/T_b)} \quad (17)$$

320 where, as usual, T_b is calculated using Eq. 10.

321 The term M, and correspondingly the M' value in the reference case is lower than in the case
 322 of all temperature-dependent properties. The difference between M' values calculated in the
 323 reference case and those in the case where all properties are temperature-dependent appreciably
 324 increase with temperature, and slightly increases with ε and with the carbon chain length of
 325 paraffin. The temperature dependence of M' given in Fig. 5b for each composite is in the
 326 temperature range from T_m to T_m+100K for its melting phase. During this temperature range,
 327 M' at different volume fraction clearly overlap. For the two paraffins here considered, the
 328 difference between M' values obtained in the reference case and that in the case considering
 329 temperature-dependence of all properties is lower than 12% at 100°C above T_m of each paraffin
 330 type. In applications where paraffins are used as thermal energy storage media these
 331 temperatures can be locally reached only when particularly high heating rates or heat flows are
 332 imposed/supplied. Under most cases the impact of temperature on viscosity prevails over the

333 other factors. In these cases, the material-dependent term becomes the ratio between the
 334 porosity and T_m dependent term F_{T_m} and the temperature-dependent term μ_T . The calculation
 335 of M_{T_m} can be further simplified considering temperature-independent properties of Al. These
 336 latter are derived from their temperature-dependent description given in [34], considering $T=$
 337 330K, i.e., an intermediate temperature of the melting temperatures for paraffins in the
 338 paraffin/Al PCM composites considered in the referred paper.

$$339 \quad M(T_m, \varepsilon) = F_{T_m} / \mu_T$$

$$340 \quad = \frac{209.5g(2452990(1-\varepsilon)+778.2(6.33 \times 10^{-3} T_m^2 - 1.03 T_m + 1961)\varepsilon)}{T_m \left(253.9(1-\varepsilon)^{1.3296} + \varepsilon \left(0.07128 \left(\frac{T_m}{T_b} \right)^2 - 0.2056 \left(\frac{T_m}{T_b} \right) + 0.2348 \right) \right)} / \left(38.78 \exp \left(-18.83 \left(\frac{T}{T_b} \right) \right) + 0.01426 \exp \left(-4.329 \left(\frac{T}{T_b} \right) \right) \right) \quad (18)$$

341 **4 Calculations of the geometry-related term in R_D equation**

342 The geometry-related term of R_D number in Eq. 2 is the product of the height H of Al/molten
 343 paraffin composite PCMs and of the permeability K of its open-cell structure. The permeability
 344 K can be substituted by the product of normalized permeability K' (a size-independent
 345 parameter [12,47]), and the square diameter of the spherical pore. Thus,

$$346 \quad K = K' \times d^2 \quad (19)$$

347 In the lattices describing the open-cell structure taken into account in the present paper (BCC,
 348 FCC, SC), a first-order correlation exists between d and the cell size L , so that K also is
 349 correlated to the square of the lattice side length L^2 .

350 Despois [47] presented a microstructure-based model to estimate the normalized permeability
 351 K' of open pore microcellular materials. Ngo and Byon [12] proposed a correlation between
 352 K' and porosity ε (ε ranging from 0.1 to 0.9) for closed and open cell simple cubic, BCC, FCC
 353 structures. To the authors' knowledge, no description of K' - ε correlation exists for the porosity
 354 values exceeding 0.90. Thus, simulated tests have been carried out to provide data for BCC
 355 structures with high ε , from which good analytical description can be derived.

356 *4.1 Direct simulations for permeability K calculation*

357 This part of the study is aimed at providing data to define the dependence of K from geometrical
 358 features in a wide range of porosity ε for an inverse BCC lattice structure. Further, the
 359 dependence or independence of geometrical features such as structure coarseness (represented
 360 by side length L) or thickness of the porous structure (represented by the number of stacked
 361 cells) is also confirmed.

362 In low velocity range of the liquid paraffin, the permeability K of a porous medium within
363 which it moves can be derived by Darcy's correlation [47], where the average velocity of the
364 liquid (u) depends on the permeability of the porous structure (K) and on the dynamic viscosity
365 μ of the paraffin and its pressure difference (ΔP) between the inlet and outlet surfaces:

$$366 \quad u = -\frac{K\Delta P}{\mu L}. \quad (20)$$

367 Thus, permeability K can be derived once u , μ and (ΔP) are known. Direct simulations are
368 performed for the unit BCC cell described in paragraph 2 and shown here in Fig.6 with the
369 boundary conditions used for these simulations in order to consider the cell as part of an
370 extremely high number of stacked cells of porous (Al) structure filled by liquid (paraffin). At
371 the inlet and outlet (on the left and right side of the cell in Fig.6, respectively), a periodic
372 pressure and velocity boundary condition is applied to simulate the situation where the fluid
373 can move across an infinite number of stacked cells in the direction of heat flow, differing only
374 by a pressure offset ΔP in the flow direction. The lateral sides are assigned as the symmetry
375 condition and the no-slip boundary condition is applied to all the solid-liquid surfaces. A steady
376 incompressible flow is assumed, and the Navier-Stokes equations are used to govern the motion
377 of fluids. All the above conditions corresponded to those previously considered by Ngo et al.
378 in [12] to the identification of permeability of porous structures with maximum pore fraction
379 of about 0.84.

380 From the simulation results, the average velocity of u (averaged on the area framed by the bold
381 red box in Fig.6) is calculated and K is derived from Eq. 20. In DS simulation, ΔP is adjusted
382 to obtain the value of Reynold number (Re) lower than 0.1, since it has been reported that Re
383 has a negligible effect on K when Re is smaller than 10^3 [12].

384 The DS model is implemented in finite element software, using a cell size L equal to 1mm.
385 Tetrahedral elements are then selected. For selected porosity levels (0.757, 0.85, 0.95), a mesh
386 sensitivity analysis has been performed, keeping element size parameters such as minimum
387 mesh size ($4 \cdot 10^{-6}$ m) and maximum element growth rate defining the maximum rate at which
388 the mesh element size can grow from one element to another (1.4) and varying the maximum
389 side length between $d/20$ and $d/100$. Mesh size-independent results are obtained with maximum
390 mesh size equal or lower than $d/80$, that is thus adopted.

391 The materials considered here for the model structure are Al and paraffin $C_{18}H_{38}$, with the
392 properties given in paragraph 3 at melting temperature (300K), even if the results on

393 permeability are not affected by the porous material as well as by the fluid filling it. Simulated
 394 tests are carried out to calculate K for several values of ε in the range 0.69-0.99. In some cases,
 395 the side length of the model is varied to 5mm to check the independence of K from L . Further,
 396 the relevance of calculated K on the PCM composite thickness is carried out by repeating
 397 selected simulations for different numbers of stacked cells in the direction of heat flow.

398 *4.2 Effect of side-length on the permeability of (inverse) BCC cells*

399 From each K , the corresponding K'_∞ (K'_∞ refers to K' obtained when a periodic boundary
 400 condition is set) is derived and plotted in Fig. 7 as a function of the porosity ε . The results show
 401 that the normalized permeability varies with porosity for BCC structure and confirm the
 402 independency of the normalized permeability from cell side, and thus from structure coarseness.

403 In the same figure the simulated tests results have been compared to experimental data. These
 404 latter have been derived for different porosity but two pore diameter values: 75 μm and 400
 405 μm . On the contrary, simulated results have been obtained for coarser structures with fixed side
 406 lengths (1 mm and 5 mm) and corresponding pore diameters increasing with the porosity. For
 407 example, in the case that side length L is fixed at 1mm, $d=925 \mu\text{m}$ for $\varepsilon=0.8$, and $d=976 \mu\text{m}$
 408 for $\varepsilon=0.9$. Pore diameters for $L=5 \text{ mm}$ are 5 times those for $L=1\text{mm}$ for a given porosity The
 409 same plot also shows the other DS results obtained for BCC structures [10,12] (unknown pore
 410 diameter or unit cell side). The agreement of all these data is clearly visible.

411 A best-fit of the above data for low permeability has been performed using the correlation K'_∞
 412 $=C*(\varepsilon-D)E$ proposed by Ngo and Byon [12] for simple cubic, BCC, FCC structures with $\varepsilon<0.90$.
 413 Since the fitting parameters are not specifically mentioned for each lattice type, but only ranges
 414 for them are given ($0.063<C<0.265$, $0.523<D<0.726$, $1.531<E< 1.994$), the authors of the
 415 present paper have identified these parameters for BCC lattice by best-fitting simulation data
 416 of normalized permeability (plotted as black line in Fig.7) for $\varepsilon < 0.90$. The following equation
 417 has been derived:

$$418 \quad K'_\infty = 0.09|\varepsilon - 0.68|^{1.6} \quad \text{for } 0.6802<\varepsilon<0.9395 \quad (21)$$

419 In Fig.7, it is observed that Eq.21 can well predict K'_∞ for the porosity up to about 0.94 and the
 420 statistical analysis shows coefficient of determination (R^2) to be 0.997 by comparing results
 421 predicted by DS and Eq.21 for $0.69<\varepsilon<0.939$. But the above analytical description of K'_∞ is

422 clearly unsuitable for the porosity over 0.94, the one which is of more interest for composite
423 PCMs.

424 Thus, the authors have used two different best fit equations, one considering the whole set of
425 simulated results, the other only for porosity exceeding 0.94. The resulting equations are the
426 following ones, and they are plotted as red and green lines in including all the simulated test
427 results.

$$428 \quad K'_{\infty} = 131.5237\varepsilon^3 - 376.2025\varepsilon^2 + 358.8131\varepsilon - 114.103 \quad \text{for } 0.9395 < \varepsilon < 0.9945 \quad (22)$$

$$429 \quad K'_{\infty} = 16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274. \quad \text{for } 0.6802 < \varepsilon < 0.9945$$

430 (23)

431 By the statistical analysis of results predicted by Eq.22 and DS method for about $0.94 < \varepsilon < 0.99$,
432 R^2 reaches 0.999. Over the entire porosity considered in Fig.7, Eq.23 can well predict K'_{∞} for
433 the porosity down to about 0.75, even though the statistical analysis show coefficient of
434 determination (R^2) to be 0.992. In the present paper, a combination of Eq.21 for low porosity
435 range and Eq.22 for high porosity range, and a single Eq.23 for a wide porosity range are both
436 applied to estimate K'_{∞} for an inverse BCC lattice structure.

437 *4.3 Effect of the height of PCM composite*

438 The previous calculations of the normalized permeability of the inverse BCC cell structure
439 consider a periodic boundary condition, and thus fluid moves across a porous structure of
440 theoretically infinite height, where no edge conditions occur. In the case of finite-thickness of
441 the porous material filled with liquid, specifically in those characterized by coarse structures,
442 the impossibility to have liquid movement across the external boundary could potentially affect
443 the convective movement of the liquid and thus different permeability could be considered.
444 The present part of the study is aimed at checking if and to what extent normalized permeability
445 is affected by specimen thickness, here considered as the number of stacked unit cells. The
446 geometrical models for DS simulations in this case include several stacked unit cells, and
447 pressure difference is set at the external inlet and outlet surface instead of periodic boundary
448 condition. The normalized permeability is derived from numerical tests results as analogously
449 to what described in paragraph 4.1. Some sets of simulated tests are carried out in the cases
450 that number of stacked cells n equal to 1, 3, 7, 9, 11, 15, 20, 30, varying the porosity between
451 0.70 and 0.96, where L is fixed at 1mm. In some cases, $L= 5\text{mm}$ is additionally considered.

452 The results of these sets are shown in Fig.8a. Here, the ratio between the calculated K'_n and
 453 K'_∞ is given in ordinate, where K'_n represents the K' calculated for n stacked cells (with given
 454 ε and L) and K'_∞ is the one calculated for the same ε and L from the simulations presented in
 455 paragraph 4.2. Datasets for different porosity values (and in some case both for $L=1$ mm and
 456 $L=5$ mm) are plotted versus the number of stacked cells. It can be easily noticed that, as
 457 expected, for high n the value of K'_n/K'_∞ tends to 1. On the other hand, porosity affects the
 458 ratio K'_1/K'_∞ , i.e. the ratio between the K' value when the thickness of the porous structure is
 459 equal to the side length of a unit cell and the K'_∞ obtained considering periodic boundary
 460 condition. As porosity increases from 0.7 to 0.96, K'_1/K'_∞ increases only from 1.0013 to
 461 1.0133. The effect of cell size is checked for a relatively critical case of 0.9, and it is found to
 462 be negligible (see Fig. 8a). Porosity further affects the number of stacked cells above which
 463 the K'_n can be considered as constant. As a matter of fact this value of n increases with ε , being
 464 close to 5 stacked cells for $\varepsilon=0.7$, 15 stacked cells for $\varepsilon=0.96$.

465 The deviation of K'_n from K'_∞ is in any case quite limited, in most cases lower than 1%. The
 466 maximum relative error considering K'_∞ instead of K'_n occurs for $n=1$ and can be easily
 467 computed for cells of the same geometry (ε and L) is calculated (in percentage) as:

$$468 \quad \delta = 100 \times |K'_\infty - K'_1| / K'_1 \quad (24)$$

469 The corresponding values for $L=1$ mm (and in some cases for $L=5$ mm) are plotted in Fig. 8b.
 470 It can be observed that δ increases with porosity. The maximum relative error δ is in any case
 471 lower than 1.4%. This means that the error committed considering the thickness-independent
 472 values and the normalized permeability model presented in paragraph 4.2 are lower than 1.4%
 473 if the ratio between the thickness of the porous medium and L is close to 1.

474 *4.4 Modelled material dependant term $K \times H$ in \mathbb{R}_D equation*

475 As a result, from the analyses of DS results performed in previous paragraphs 4.2 and 4.3, the
 476 geometry-only dependent term to be inserted in the equation to calculate the \mathbb{R}_D number can be
 477 well approximated by following analytical descriptions:

$$478 \quad KH = K'_\infty \times d^2 \times H \quad (25)$$

479 While Eq. 21-23 supply analytical descriptions in terms of $K'_\infty = K'_\infty(\varepsilon)$ for different porosity
 480 ε ranges. On the basis of the geometry relationship $d/L=f(\varepsilon)$ for inverse BCC cell as discussed
 481 previously, the equation (25) can be written in terms of H , porosity ε , side length L of the BCC

482 lattice. The description of the geometry-dependent term in \mathbb{R}_D number is here presented in
 483 terms of the length L of the inverse BCC structure considering analytical descriptions Eq.23
 484 for K'_∞ and Eq.8 for ratio d/L:

$$485 \quad KH = K'_\infty d^2 H = (16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274) \times$$

$$486 \quad (4.969\varepsilon^3 - 11.683\varepsilon^2 + 9.599\varepsilon - 1.825)^2 L^2 H \quad (26)$$

487 Further, many available geometrical data for open cell porous structures are supplied
 488 characterizing the material coarseness by means of the parameter pores per inch (PPI). When
 489 an inverse BCC structure is considered, the correlation between L and PPI is the following [11]:

$$490 \quad L [mm] = 25.4/PPI. \quad (27)$$

491 The geometry-dependent term for \mathbb{R}_D number calculation can thus be expressed in terms of PPI
 492 and ε (to be used in the range as $0.75 < \varepsilon < 0.96$) as:

$$493 \quad KH = (16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274) \times (4.969 * \varepsilon^3 -$$

$$494 \quad 11.683 * \varepsilon^2 + 9.599 * \varepsilon - 1.825)^2 \times (0.0254/PPI)^2 \times H \quad (28)$$

495 **5 Modelled \mathbb{R}_D number for Al/Paraffin PCM**

496 For a PCM composite formed by an Al porous structure and a paraffin with melting temperature
 497 in the range 280-340K, \mathbb{R}_D number can be calculated by combining Eq. 15 with Eq. 26 (or ,
 498 alternatively, 28), as follows:

$$499 \quad \mathbb{R}_D =$$

$$500 \quad \frac{209.5g((\rho C_p)_{Met}(1-\varepsilon) + (\rho C_p)_{paraffin}\varepsilon)(16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274)(4.969\varepsilon^3 - 11.683\varepsilon^2 + 9.599\varepsilon - 1.825)^2 \Delta T H L^2}{T_m * (\lambda_{Met} (1-\varepsilon)^{1.3296} + \varepsilon \lambda_{paraffin}) * \left(38.78 \times \exp\left(-18.83\left(\frac{T}{T_b}\right)\right) + 0.01426 \times \exp\left(-4.329\left(\frac{T}{T_b}\right)\right) \right)}$$

$$501 \quad (29)$$

502 A simplified and approximated equation considered constant-Al phase properties, and density,
 503 C_p and thermal conductivity of paraffin at its melting temperature can be obtained combining
 504 Eq. (18) with Eq. 26 (or, alternatively, 28).

$$505 \quad \mathbb{R}_D = \frac{209.5g(2452990(1-\varepsilon) + 778.2(6.33 \times 10^{-3} T_m^2 - 1.03 T_m + 1961)\varepsilon)(16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274) * (4.969\varepsilon^3 - 11.683\varepsilon^2 + 9.599\varepsilon - 1.825)^2 \Delta T H L^2}{T_m * \left(253.9(1-\varepsilon)^{1.3296} + \varepsilon * \left(0.07128 \times \left(\frac{T_m}{T_b}\right)^2 - 0.2056 \times \left(\frac{T_m}{T_b}\right) + 0.2348 \right) \right) \left(38.78 \times \exp\left(-18.83\left(\frac{T}{T_b}\right)\right) + 0.01426 \times \exp\left(-4.329\left(\frac{T}{T_b}\right)\right) \right)}$$

$$506 \quad (30)$$

507 where, as usual, T_b is calculated using Eq. 10.

508 The above equations supply good description of \mathbb{R}_D for the porosity range $0.75 < \varepsilon < 0.9945$ since
 509 Eq.26 is based on Eq.23 well predicting K'_∞ for the porosity larger than 0.75. For the same
 510 porosity range, similar equations can be written combining Eq. 15 or 18 with Eq. 28, when the
 511 structure coarseness of the Al porous structure is given in terms of PPI.

512 5.1 Conditions for conduction-dominant heat transport

513 Once \mathbb{R}_D is modelled, it is possible to calculate if the conditions for conduction-dominant heat
 514 transport are satisfied or not (and natural convection occurs) for a specific Al/paraffin PCM
 515 composite. As mentioned in the introduction, natural convection for the liquid paraffin can
 516 occur as \mathbb{R}_D overcomes critical \mathbb{R}_D values, here referred as \mathbb{R}_{DC} . \mathbb{R}_{DC} is related to heating
 517 location. Specifically, critical \mathbb{R}_D values \mathbb{R}_{DCI} is about 40 for heat supplied from the horizontal
 518 bottom surface of the composite material [31,33] and $\mathbb{R}_{DCV} = 1$ can be considered for heat
 519 supplied horizontally from one lateral vertical surface of the PCM composites [32].

520 The conditions where conduction-dominant heat transport occurs can be thus obtained focusing
 521 on several geometrical or material parameters (such as the cell coarseness L, the porosity ε , the
 522 specimen thickness H), by rearranging Eq.29 (or its simplified version, Eq.30). A similar
 523 approach could be followed by considering the selection of different paraffins, characterising
 524 them in terms of their melting temperature T_m . The above equations are the ones valid for the
 525 wider porosity range. Better description for high- and low- porosity ranges can be obtained by
 526 using the corresponding analytical description of K'_∞ (i.e., Eq.21 and Eq.22) into the geometry-
 527 dependent term for \mathbb{R}_D estimation.

528 For the case of length, the condition for conduction-dominant cases is when L is lower than the
 529 critical value L_c , the most general equation is:

$$530 L < L_c =$$

$$531 \sqrt{\frac{\mathbb{R}_{DC} T_m (\lambda_{Met} (1-\varepsilon)^{1.3296} + \varepsilon \lambda_{paraffin}) \times \left(38.78 \times \exp\left(-18.83 \frac{T}{T_b}\right) + 0.01426 \times \exp\left(-4.329 \frac{T}{T_b}\right) \right)}{209.5g((\rho C_p)_{Met}(1-\varepsilon) + (\rho C_p)_{paraffin}\varepsilon)(16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274)(4.969\varepsilon^3 - 11.683\varepsilon^2 + 9.599\varepsilon - 1.825)^2}} \left(\frac{1}{\Delta TH} \right) \quad (31)$$

532 While in the case where the thermophysical parameters (λ , ρ and C_p) of Al are constant and
 533 those of paraffin are values at its melting temperature, the equation becomes:

$$534 L = L_c <$$

$$535 \sqrt{\frac{\mathbb{R}_{DC} T_m (253.9(1-\varepsilon)^{1.3296} + \varepsilon \left(0.07128 \times \left(\frac{T_m}{T_b} \right)^2 - 0.2056 \times \left(\frac{T_m}{T_b} \right) + 0.2348 \right)) \times \left(38.78 \times \exp\left(-18.83 \frac{T}{T_b}\right) + 0.01426 \times \exp\left(-4.329 \frac{T}{T_b}\right) \right)}{2055.195(2452990(1-\varepsilon) + 778.2(6.33 \times 10^{-3} T_m^2 - 1.03 T_m + 1961)\varepsilon)(16.0594\varepsilon^4 - 52.4791\varepsilon^3 + 64.1654\varepsilon^2 - 34.7482\varepsilon + 7.0274)(4.969\varepsilon^3 - 11.683\varepsilon^2 + 9.599\varepsilon - 1.825)^2}} \left(\frac{1}{\Delta TH} \right)$$

$$536 (32)$$

537 Where, as usual, T_m and T_b (derived by Eq. 10) and ΔT are given in K, H and L are given in m,
538 and ε (porosity) corresponds to the volume fraction of the paraffins used as PCM.

539 A reference condition is considered for the case of horizontal heating ($R_{DC}=1$), the low-melting
540 paraffin $C_{18}H_{38}$ ($T_m=300$ K), $H=25$ mm and $\Delta T =10$ K, with the surface opposite to the heated
541 one at T_m . The correlation between the critical length L_c and the porosity ε can be derived from
542 the simplified Eq. 32, considering the temperature T (to estimate dynamic viscosity at $T_m+\Delta T$)
543 and plotted as black lines in Fig. 9a-9c. For comparison purposes, lines corresponding to data
544 calculated using the high- and low porosity equations (Eq.21 and Eq.22) for K'_∞ are drawn as
545 dashed lines. These three figures further represent the case where the paraffin types, the height
546 of PCM composite, or the temperature difference across it are varied, respectively. Fig. 9a
547 shows that, for relatively high porosity range (ε above 0.9) and for the thickness of 25 mm and
548 $\Delta T =10$ K, there is no convection motion for all the considered paraffin if the length of the unit
549 cell is smaller than 1.3 mm (about 20 PPI), to be read in the right-side vertical axis. It is also
550 clear that the approximated description of K'_∞ does not predict good L_c for porosity lower than
551 0.75, where K'_∞ is low and the discrepancy between its values obtained using single- or two-
552 range for ε is relatively high. On the contrary, for the low porosity porous material (i.e., lower
553 than 0.75), where the molten filling material can be considered as almost trapped into isolated
554 spheres, the structure coarseness range where the heat transfer is conductive increases a lot (up
555 to $L=6$ mm, corresponding to less than about 4 PPI). The direct correlation of L_c ($1/H$) is clearly
556 visible in Fig. 9b, where reduces for an increase of H from 5 mm to 50 mm.

557 The effect of ΔT , shown in Fig. 9c, is similar to that of H since the correlation between L_c and
558 H or ΔT is the same. It can be further observed that Eq. 32 can be rearranged considering the
559 average temperature gradient $\Delta T/H$ (assumed to be constant along the thickness) instead of the
560 temperature difference ΔT . The results from part of the second and third set data (given in Fig.
561 9b and 9c) are presented in terms of the average temperature gradients $\Delta T/H$ in Fig.9d, only
562 considering the single-range simplified Eq. 32 for porosity values exceeding 0.75.

563 The reference condition used to draw plots in Figure 9 considers the specimens heated on a
564 lateral surface, thus R_{DC} equals to 1. Similar evaluations of the combined effects of porosity,
565 paraffin type and ΔT could be obtained for different heating by inserting proper R_{DC} (such as
566 the abovementioned value close to 40 in the case of specimens heated from the bottom).

567 *5.2 Validation of the simplified model for R_D number calculation*

568 Experimental data available from literature [25] have been used to validate the model with
569 single-porosity range description of K'_{∞} (Eq. 32). These values refer to composite PCMs made
570 by pure Al foams (their stated thermal conductivity is 236 W/(m·K)) which have been filled
571 by a paraffin stated to melt at about 300K (T_m close to that of the paraffin $C_{18}H_{38}$). Two sets of
572 composite PCM samples, all having the same size 101.6 mm \times 101.6 mm \times 50.8 mm, confined
573 laterally by insulated plates. The first set is characterized by the same pore coarseness (10 PPI)
574 and different porosity (0.884, 0.934, 0.966). The second set is characterized by a constant
575 porosity of about 0.936 and different PPI (10, 20, 40). The samples were heated with constant
576 power on their square surface, placed vertically and the temperature at several location from
577 the heated to the opposite surfaces were measured and the temperature profiles were given in
578 the paper. Thus, the model was applied here considering R_{DC} equal to 1. The possibility to
579 observe the melting interface, due to the paraffin feature changing from opaque to transparent
580 as melting occurs was used by Lafdi et al. [25]. The representative results of the tests [25] are
581 shown in Fig.10. Among the experimental specimens, the first one shown here exhibits the
582 strong natural convection. The specimen, when heater is on the left surface, where the
583 solid/liquid profile inclined rightward in its upper part means that convection is taking place.

584 In order to carry out the validation of the model described by Eq. 32, the identification of the
585 temperatures and H is needed for each specimen. The temperature on the vertical, heated
586 surface is derived from temperature profiles at the (different) time at which macrographs of the
587 samples were taken by Lafdi et al. [25]. This temperature is used to calculate ΔT as the
588 difference between it and T_m . In the experimental tests, the previously described specimen
589 thickness cannot be considered as the one used to derive L_c - ε correlation, since the molten
590 liquid does not fill the whole thickness. Thus, for the 6 specimens for which the solid/liquid
591 interfaces are available, H is considered as the distance between the external heating surface
592 and the solid/liquid interface at the height level where the profile changes from vertical to
593 inclined (see examples in Fig. 10). The H values derived in this way, as well as the ΔT are
594 given for each experimental test in the legend of Fig. 11, where experimental points
595 corresponding to each specimen are placed in terms of their PPI and porosity. Further, Fig. 11
596 shows the curves for L_c calculated on the basis of H and the ΔT for each sample, adopting the
597 same colour. Thus, for the two specimens with the same porous structure characteristics (10
598 PPI and porosity 0.934) two L_c curves are derived, each corresponding to an experimental test.
599 The presence of convection in each test can be compared to the position of the corresponding
600 line. Most of the points lay above the corresponding curves, and convection is actually

601 observed by Lafdi et al. [25]. For them, the solid-liquid interface is more inclined as the relative
602 distance between points and the corresponding line increases. Convection motion is predicted
603 and experimentally demonstrated not to occur for the sample of set 1 with the lowest porosity
604 (black point and line). Lastly, for the sample of set 2 with the highest PPI (lowest L), the point
605 is only slightly above the corresponding L_c line, meaning convection motion is quite weak.
606 Also, this situation corresponds to experimental observations.

607 As a result, the experimental tests by Lafdi et al [25] validate the model for calculation of
608 conditions leading onset of convection (or, alternatively, for conduction-dominant heat transfer)
609 for Al/paraffin composites.

610 **6 Conclusion**

611 The composite PCMs considered in the present paper are formed by paraffins with open-cell
612 metallic Al porous foam which has been modelled as a (inverse) body-centred cubic (BCC)
613 lattice. Analytical expressions have been given to correlate porosity with side length, pore
614 diameter or pores per inch for this structure.

615 Further, analytical expression has been proposed to model the effective thermal conductivity
616 of the composite PCMs, indirectly based on the melting temperature of the PCM. The BCC
617 structure has been proved to be the most suitable to model this property for Al foams with high
618 porosity.

619 The Rayleigh Darcy (R_D) number can be obtained as a product of the temperature difference,
620 the material-dependent term M and the geometry-dependent term (KH). M has been
621 analytically described by a general version or a simplified one, and (KH) also has been
622 analytically calculated for different porosity ranges for composite PCMs. With respect to
623 previous analytical models for permeability for porosity less than 0.94, the description adopted
624 here has been proved to be suitable also for porosity in the 0.94-0.99 range.

625 The above equations are used to calculate critical conditions for conduction-dominant heat
626 transfer of the paraffin within a metallic foam when heated horizontally. Specifically, the model
627 has been rearranged to predict the porous structure coarseness (in terms of L or PPI) leading to
628 natural convection for the different porosity, thickness of specimen containing molten paraffin,
629 its temperature range and paraffin types. A graphical procedure is suggested to check the onset
630 of natural convection, which is adopted to validate the simplified model by the experimental

631 results for several Al open-cell foams filled with specific paraffin. They have a good
632 agreement.

633 The supplied equations and plots for critical conditions derived for horizontal heating can be
634 easily modified for the case of heating from the bottom of the material (in this case L_c is about
635 6.3 times the one presented in this paper). The approach used here can be further extended to
636 different composite materials containing a liquid phase and an open-cell porous structure, with
637 a simple definition of M term.

638 **Acknowledgements**

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642 **Reference**

- 643 [1] K. Pielichowska, K. Pielichowski, Phase change materials for thermal energy storage,
644 *Prog. Mater. Sci.* 65 (2014) 67–123. doi:10.1016/j.pmatsci.2014.03.005.
- 645 [2] H. Nazir, M. Batool, F.J. Bolivar Osorio, M. Isaza-Ruiz, X. Xu, K. Vignarooban, P.
646 Phelan, Inamuddin, A.M. Kannan, Recent developments in phase change materials for
647 energy storage applications: A review, *Int. J. Heat Mass Transf.* 129 (2019) 491–523.
648 doi:10.1016/j.ijheatmasstransfer.2018.09.126.
- 649 [3] P. Sobolciak, M. Karkri, M.A. Al-Maadeed, I. Krupa, Thermal characterization of
650 phase change materials based on linear low-density polyethylene, paraffin wax and
651 expanded graphite, *Renew. Energy.* 88 (2016) 372–382.
652 doi:10.1016/j.renene.2015.11.056.
- 653 [4] G. Wang, G. Wei, C. Xu, X. Ju, Y. Yang, X. Du, Numerical simulation of effective
654 thermal conductivity and pore-scale melting process of PCMs in foam metals, *Appl.*
655 *Therm. Eng.* 147 (2019) 464–472. doi:10.1016/j.applthermaleng.2018.10.106.
- 656 [5] H. JianShe, Y. Chao, Z. Xu, Z. Jiao, D. JinXing, Structure and thermal properties of
657 expanded graphite/paraffin composite phase change material, *Energy Sources, Part A*
658 *Recover. Util. Environ. Eff.* 41 (2019) 86–93. doi:10.1080/15567036.2018.1496199.
- 659 [6] W. Li, Y. Dong, X. Zhang, X. Liu, Preparation and Performance Analysis of Graphite
660 Additive/Paraffin Composite Phase Change Materials, *Processes.* 7 (2019) 447.

- 661 doi:10.3390/pr7070447.
- 662 [7] Y. Yao, H. Wu, Thermal transport process of metal foam/paraffin composite (MFPC)
663 with solid-liquid phase change: An experimental study, *Appl. Therm. Eng.* 179 (2020)
664 115668. doi:10.1016/j.applthermaleng.2020.115668.
- 665 [8] Z. Tao, H. Wang, J. Liu, W. Zhao, Z. Liu, Q. Guo, Dual-level packaged phase change
666 materials – thermal conductivity and mechanical properties, *Sol. Energy Mater. Sol.*
667 *Cells.* 169 (2017) 222–225. doi:10.1016/j.solmat.2017.05.030.
- 668 [9] W. Xu, H. Zhang, Z. Yang, J. Zhang, The effective thermal conductivity of three-
669 dimensional reticulated foam materials, *J. Porous Mater.* 16 (2009) 65–71.
670 doi:10.1007/s10934-007-9169-4.
- 671 [10] S. Krishnan, J.Y. Murthy, S. V. Garimella, Direct Simulation of Transport in Open-
672 Cell Metal Foam, *J. Heat Transfer.* 128 (2006) 793–799. doi:10.1115/1.2227038.
- 673 [11] L. Zhang, X. Zhang, Z. Liu, An Efficient Numerical Method for Pressure Loss
674 Investigation in an Oil/Air Separator with Metal Foam in an Aero-Engine, *Energies.* 13
675 (2020) 346. doi:10.3390/en13020346.
- 676 [12] I.-L. Ngo, C. Byon, Permeability of microporous wicks with geometric inverse to
677 sintered particles, *Int. J. Heat Mass Transf.* 92 (2016) 298–302.
678 doi:10.1016/j.ijheatmasstransfer.2015.08.040.
- 679 [13] X. Sun, Q. Zhang, M.A. Medina, K.O. Lee, Experimental observations on the heat
680 transfer enhancement caused by natural convection during melting of solid–liquid
681 phase change materials (PCMs), *Appl. Energy.* 162 (2016) 1453–1461.
682 doi:10.1016/j.apenergy.2015.03.078.
- 683 [14] P. Ternik, R. Rudolf, Heat transfer enhancement for natural convection flow of water-
684 based nanofluids in a square enclosure, *Int. J. Simul. Model.* 11 (2012) 29–39.
685 doi:10.2507/IJSIMM11(1)3.198.
- 686 [15] R. Pakrouh, M.J. Hosseini, A.A. Ranjbar, A parametric investigation of a PCM-based
687 pin fin heat sink, *Mech. Sci.* 6 (2015) 65–73. doi:10.5194/ms-6-65-2015.
- 688 [16] P.T. Sardari, H.I. Mohammed, D. Giddings, G.S. Walker, M. Gillott, D. Grant,
689 Numerical study of a multiple-segment metal foam-PCM latent heat storage unit:
690 Effect of porosity, pore density and location of heat source, *Energy.* 189 (2019)

- 691 116108. doi:10.1016/j.energy.2019.116108.
- 692 [17] A.A. Rabienataj Darzi, M. Jourabian, M. Farhadi, Melting and solidification of PCM
693 enhanced by radial conductive fins and nanoparticles in cylindrical annulus, *Energy*
694 *Convers. Manag.* 118 (2016) 253–263. doi:10.1016/j.enconman.2016.04.016.
- 695 [18] J. Xie, K.F. Choo, J. Xiang, H.M. Lee, Characterization of natural convection in a
696 PCM-based heat sink with novel conductive structures, *Int. Commun. Heat Mass*
697 *Transf.* 108 (2019) 104306. doi:10.1016/j.icheatmasstransfer.2019.104306.
- 698 [19] M. Ghalambaz, A.J. Chamkha, D. Wen, Natural convective flow and heat transfer of
699 Nano-Encapsulated Phase Change Materials (NEPCMs) in a cavity, *Int. J. Heat Mass*
700 *Transf.* (2019). doi:10.1016/j.ijheatmasstransfer.2019.04.037.
- 701 [20] M. Ghalambaz, S.A.M. Mehryan, I. Zahmatkesh, A. Chamkha, Free convection heat
702 transfer analysis of a suspension of nano-encapsulated phase change materials
703 (NEPCMs) in an inclined porous cavity, *Int. J. Therm. Sci.* (2020).
704 doi:10.1016/j.ijthermalsci.2020.106503.
- 705 [21] H. Zheng, C. Wang, Q. Liu, Z. Tian, X. Fan, Thermal performance of copper
706 foam/paraffin composite phase change material, *Energy Convers. Manag.* 157 (2018)
707 372–381. doi:10.1016/j.enconman.2017.12.023.
- 708 [22] Y. Tian, C.Y. Zhao, A numerical investigation of heat transfer in phase change
709 materials (PCMs) embedded in porous metals, *Energy.* 36 (2011) 5539–5546.
710 doi:10.1016/j.energy.2011.07.019.
- 711 [23] X. Hu, H. Wan, S.S. Patnaik, Numerical modeling of heat transfer in open-cell micro-
712 foam with phase change material, *Int. J. Heat Mass Transf.* 88 (2015) 617–626.
713 doi:10.1016/j.ijheatmasstransfer.2015.04.044.
- 714 [24] S.M. Hashem Zadeh, S.A.M. Mehryan, M. Ghalambaz, M. Ghodrat, J. Young, A.
715 Chamkha, Hybrid thermal performance enhancement of a circular latent heat storage
716 system by utilizing partially filled copper foam and Cu/GO nano-additives, *Energy.*
717 213 (2020) 118761. doi:10.1016/j.energy.2020.118761.
- 718 [25] K. Lafdi, O. Mesalhy, S. Shaikh, Experimental study on the influence of foam porosity
719 and pore size on the melting of phase change materials, *J. Appl. Phys.* 102 (2007)
720 083549. doi:10.1063/1.2802183.

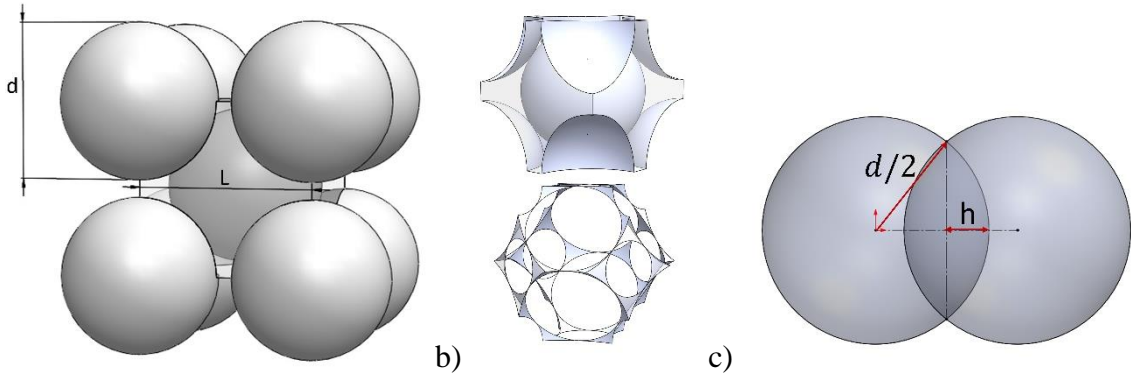
- 721 [26] M. Caliano, N. Bianco, G. Graditi, L. Mongibello, Analysis of a phase change
722 material-based unit and of an aluminum foam/phase change material composite-based
723 unit for cold thermal energy storage by numerical simulation, *Appl. Energy*. 256
724 (2019) 113921. doi:10.1016/j.apenergy.2019.113921.
- 725 [27] B. Buonomo, H. Celik, D. Ercole, O. Manca, M. Mobedi, Numerical study on latent
726 thermal energy storage systems with aluminum foam in local thermal equilibrium,
727 *Appl. Therm. Eng.* 159 (2019) 113980. doi:10.1016/j.applthermaleng.2019.113980.
- 728 [28] H.M. Sadeghi, M. Babayan, A. Chamkha, Investigation of using multi-layer PCMs in
729 the tubular heat exchanger with periodic heat transfer boundary condition, *Int. J. Heat*
730 *Mass Transf.* (2020). doi:10.1016/j.ijheatmasstransfer.2019.118970.
- 731 [29] A. Chamkha, A. Veismoradi, M. Ghalambaz, P. Talebizadehsardari, Phase change heat
732 transfer in an L-shape heatsink occupied with paraffin-copper metal foam, *Appl.*
733 *Therm. Eng.* (2020). doi:10.1016/j.applthermaleng.2020.115493.
- 734 [30] A.I. Alsabery, R. Mohebbi, A.J. Chamkha, I. Hashim, Effect of local thermal non-
735 equilibrium model on natural convection in a nanofluid-filled wavy-walled porous
736 cavity containing inner solid cylinder, *Chem. Eng. Sci.* (2019).
737 doi:10.1016/j.ces.2019.03.006.
- 738 [31] E.B. Soboleva, Rayleigh-Darcy convection in a porous layer: A comparison of near-
739 critical and normal fluid phases, *ArXiv Prepr. ArXiv1001.4139*. (2010).
- 740 [32] D. Poulikakos, Unsteady natural convection in a porous layer, *Phys. Fluids*. 26 (1983)
741 1183. doi:10.1063/1.864266.
- 742 [33] L.E. Howle, Convection in Ordered and Disordered Porous Layers, in: D.B. Ingham,
743 I.B.T.-T.P. in P.M.I.I. Pop (Eds.), *Transp. Phenom. Porous Media II*, Elsevier, Oxford,
744 2002: pp. 155–176. doi:10.1016/B978-008043965-5/50007-6.
- 745 [34] Z. Li, E. Gariboldi, Review on the temperature-dependent thermophysical properties of
746 liquid paraffins and composite PCMs with metallic porous structures, *Mater. Today*
747 *Energy*. (2021) 100642. doi:10.1016/j.mtener.2021.100642.
- 748 [35] T. Fiedler, I.V. Belova, A. Rawson, G.E. Murch, Optimized Lattice Monte Carlo for
749 thermal analysis of composites, *Comput. Mater. Sci.* 95 (2014) 207–212.
750 doi:10.1016/j.commatsci.2014.07.029.

- 751 [36] E. Solórzano, J.A. Reglero, M.A. Rodríguez-Pérez, D. Lehmhus, M. Wichmann, J.A.
752 de Saja, An experimental study on the thermal conductivity of aluminium foams by
753 using the transient plane source method, *Int. J. Heat Mass Transf.* 51 (2008) 6259–
754 6267. doi:10.1016/j.ijheatmasstransfer.2007.11.062.
- 755 [37] A. Bhattacharya, V.V. Calmidi, R.L. Mahajan, Thermophysical properties of high
756 porosity metal foams, *Int. J. Heat Mass Transf.* 45 (2002) 1017–1031.
757 doi:10.1016/S0017-9310(01)00220-4.
- 758 [38] R. Singh, H.. Kasana, Computational aspects of effective thermal conductivity of
759 highly porous metal foams, *Appl. Therm. Eng.* 24 (2004) 1841–1849.
760 doi:10.1016/j.applthermaleng.2003.12.011.
- 761 [39] K. Boomsma, D. Poulikakos, On the effective thermal conductivity of a three-
762 dimensionally structured fluid-saturated metal foam, *Int. J. Heat Mass Transf.* 44
763 (2001) 827–836. doi:10.1016/S0017-9310(00)00123-X.
- 764 [40] V. V. Calmidi, R.L. Mahajan, The Effective Thermal Conductivity of High Porosity
765 Fibrous Metal Foams, *J. Heat Transfer.* 121 (1999) 466–471. doi:10.1115/1.2826001.
- 766 [41] X.H. Yang, J.J. Kuang, T.J. Lu, F.S. Han, T. Kim, A simplistic analytical unit cell
767 based model for the effective thermal conductivity of high porosity open-cell metal
768 foams, *J. Phys. D. Appl. Phys.* 46 (2013) 255302. doi:10.1088/0022-
769 3727/46/25/255302.
- 770 [42] A.J. Rawson, E. Kisi, C. Wensrich, Microstructural efficiency: Structured
771 morphologies, *Int. J. Heat Mass Transf.* 81 (2015) 820–828.
772 doi:10.1016/j.ijheatmasstransfer.2014.11.012.
- 773 [43] X. Xiao, P. Zhang, M. Li, Effective thermal conductivity of open-cell metal foams
774 impregnated with pure paraffin for latent heat storage, *Int. J. Therm. Sci.* 81 (2014)
775 94–105. doi:10.1016/j.ijthermalsci.2014.03.006.
- 776 [44] Y. Yao, H. Wu, Z. Liu, A new prediction model for the effective thermal conductivity
777 of high porosity open-cell metal foams, *Int. J. Therm. Sci.* 97 (2015) 56–67.
778 doi:10.1016/j.ijthermalsci.2015.06.008.
- 779 [45] M.M. EL IDI, M. KARKRI, M. KRAIEM, Preparation and effective thermal
780 conductivity of a Paraffin/ Metal Foam composite, *J. Energy Storage.* 33 (2021)

- 781 102077. doi:10.1016/j.est.2020.102077.
- 782 [46] R.C. Progelhof, J.L. Throne, R.R. Ruetsch, Methods for predicting the thermal
783 conductivity of composite systems: A review, *Polym. Eng. Sci.* 16 (1976) 615–625.
784 doi:10.1002/pen.760160905.
- 785 [47] J. DESPOIS, A. MORTENSEN, Permeability of open-pore microcellular materials,
786 *Acta Mater.* 53 (2005) 1381–1388. doi:10.1016/j.actamat.2004.11.031.
- 787

Nomenclature			
d	Diameter of sphere [m]	ε	Porosity [-]
g	Gravitational acceleration [m/s^2]	θ	a fitting constant of Progelhof model [-]
H	Specimen height [m]	λ	thermal conductivity [$\text{W}/(\text{m}\cdot\text{K})$]
I	Thermal inertia [$\text{kg}/(\text{s}^2\cdot\text{m}\cdot\text{K})$]	Abbreviation/subscripts	
K	Permeability [m^2]	∞	infinite value
K'	Normalized permeability [-]	1	one unit cell
L	Side length [m]	\perp	heat supplied from the horizontal bottom surface
L_c	Critical side length [m] for onset of natural convection	BCC	body-centered cubic
M	Material dependent term of RD	c	critical value
M'	Ratio between M at a given temperature and the value of M assumed at the melting temperature of the molten (PCM, paraffin) phase	DS	direct simulation
n	Number of stacked cells	eff	effective value
P	Pressure [Pa]	FCC	face-centred cubic
R^2	Coefficient of determination [-]	high	high-conductivity phase
Re	Reynold number [-]	j	j^{th} phase
T	Temperature [K]	LMC	lattice Monte Carlo
Tb	Boiling temperature [K]	low	low-conductivity phase
Greek symbols		Met	metallic phase of composites

μ	Dynamic viscosity [Pa·s]	NETC	normalized effective thermal conductivity
ρ	Density [kg/m ³]	PCM	phase change material
u	Average velocity of the liquid [m/s]	PPI	pores per inch
α	Thermal diffusivity [m ² /s]	R_D	Rayleigh-Darcy number
β	Coefficient of volumetric thermal expansion [1/K]	SC	simple cubic
δ	maximum relative error considering K'_{\forall} instead of K'_1 [-]	TES	thermal energy storage
Δ	Difference	F	heat supplied horizontally from one lateral vertical surface

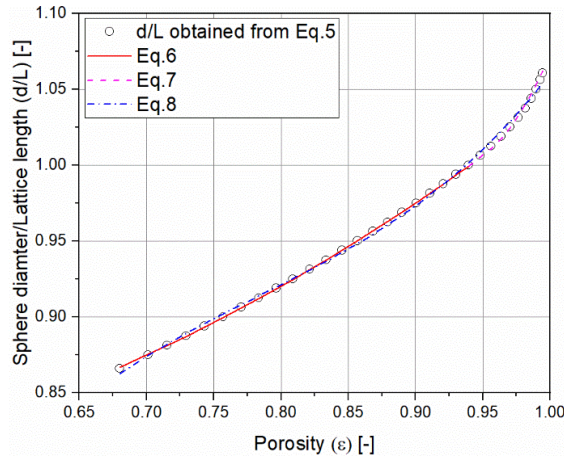


788 a)

b)

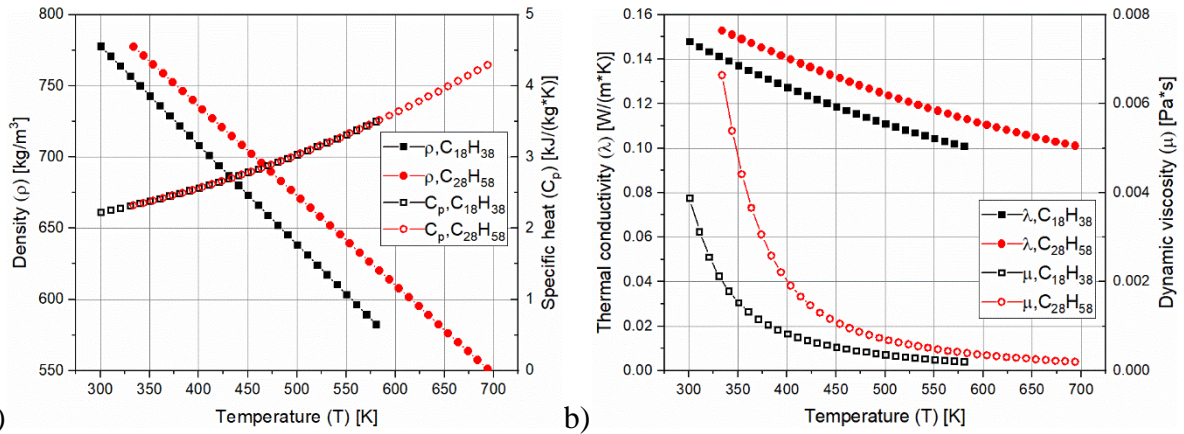
c)

789 Figure 1 a) Schematic representation of the interconnected spheres in a BCC structure (volume
790 filled by the PCM); b) Al unit inverse BCC cell for the two extreme cases; c) geometrical
791 definition of h.



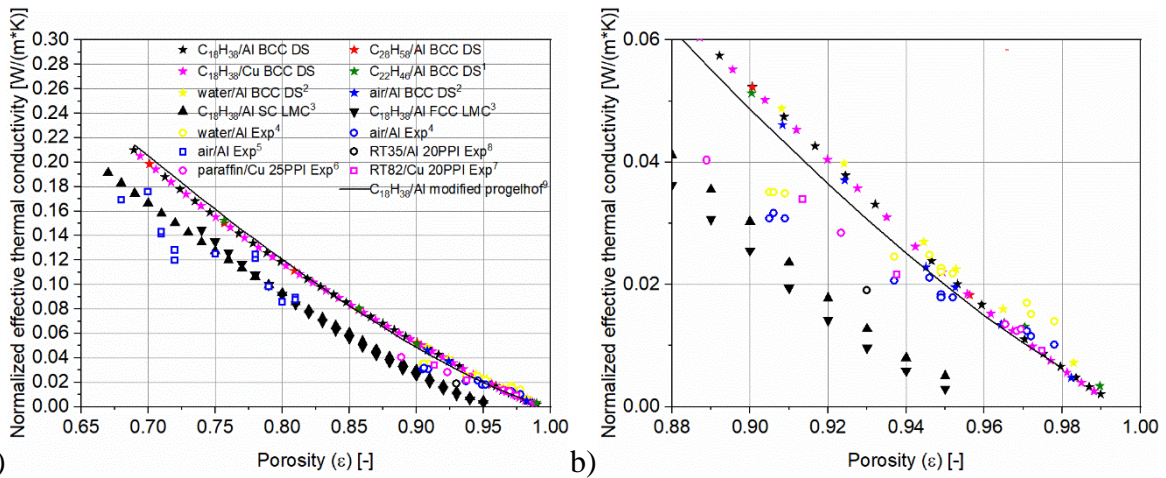
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793 Figure 2 Correlation between porosity ϵ in (inverse) BCC structure and the sphere diameter to
 794 lattice length ratio (d/L). Datapoints refer to data derived from Eq.5, while curves refer to the
 795 best fit equations in different porosity ranges (Eq.6-8).



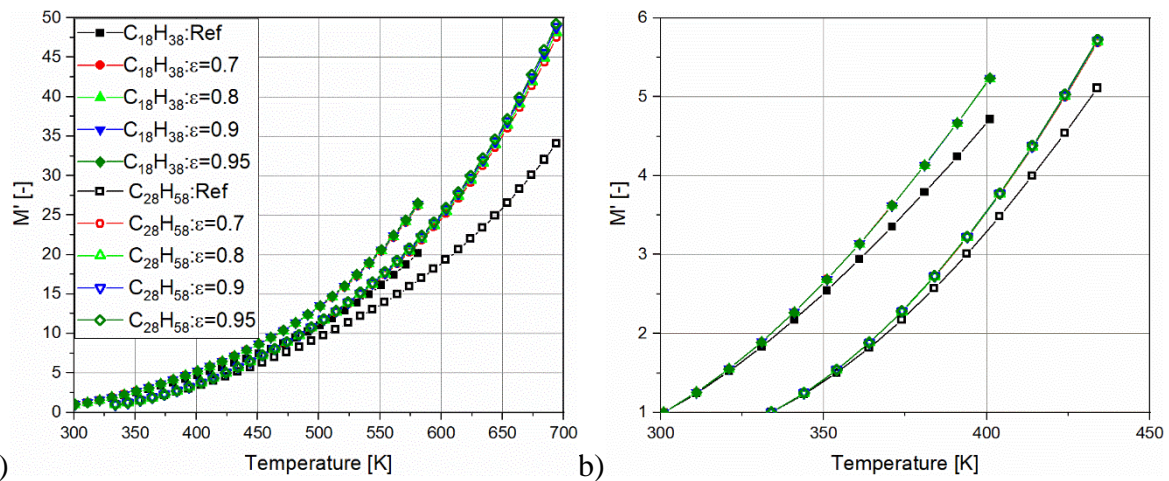
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797 Figure 3 Temperature-dependence of thermophysical properties of paraffin $C_{18}H_{38}$ and $C_{28}H_{58}$.
 798 a) Density and specific heat; b) Thermal conductivity and dynamic viscosity

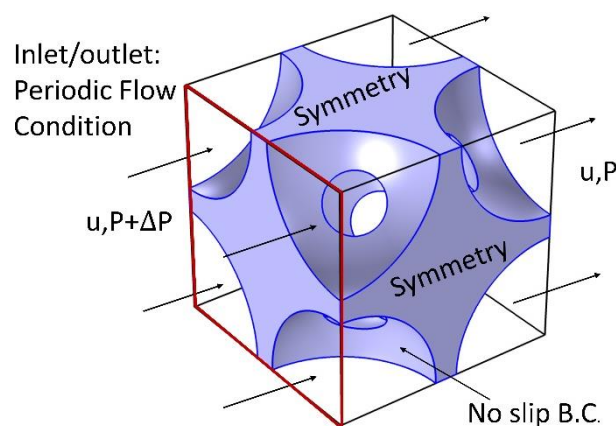


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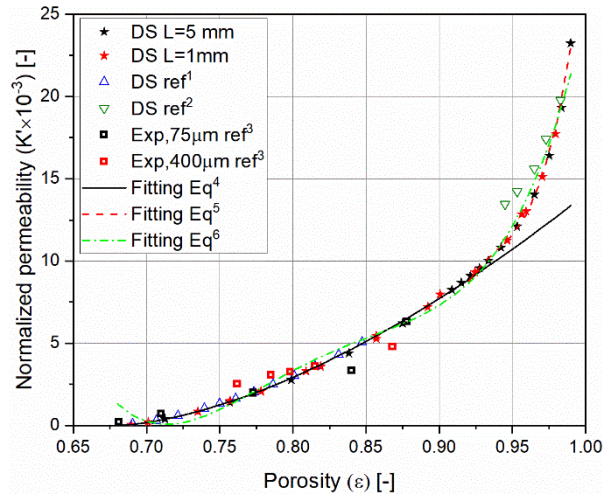
800 Figure 4 Normalized effective thermal conductivity (NETC) of composite structure with an
 801 open-cell porous structure at different porosity (volume fraction) ε . Fig. 4b illustrates the data
 802 of Fig. 4a in the high porosity- low NETC range. The combinations of paraffins, air or water
 803 and the metallic structures used are given in the legend (the same for both plots). The literature
 804 works where data have been obtained can be derived from apex numbers as follows: 1 refers
 805 to [23], 2 is [10], 3 is [42], 4 is [40], 5 is [36], 6 is [43], 7 is [44], 8 is [45] 9 is [34].



807 Figure 5 a) Temperature-dependence of material-dependent term M' for composite PCMs
 808 formed by paraffins $C_{18}H_{38}$ or $C_{28}H_{58}$ and Al porous structure with porosity in the range 0.7-
 809 0.95. Ref corresponds to the case of M' calculated considering all properties constant at T_m
 810 with the only temperature-dependence associated with the dynamic viscosity b) Temperature-
 811 dependence of the M' in the temperature range from T_m to T_m+100K for $C_{18}H_{38}$ or $C_{28}H_{58}$.

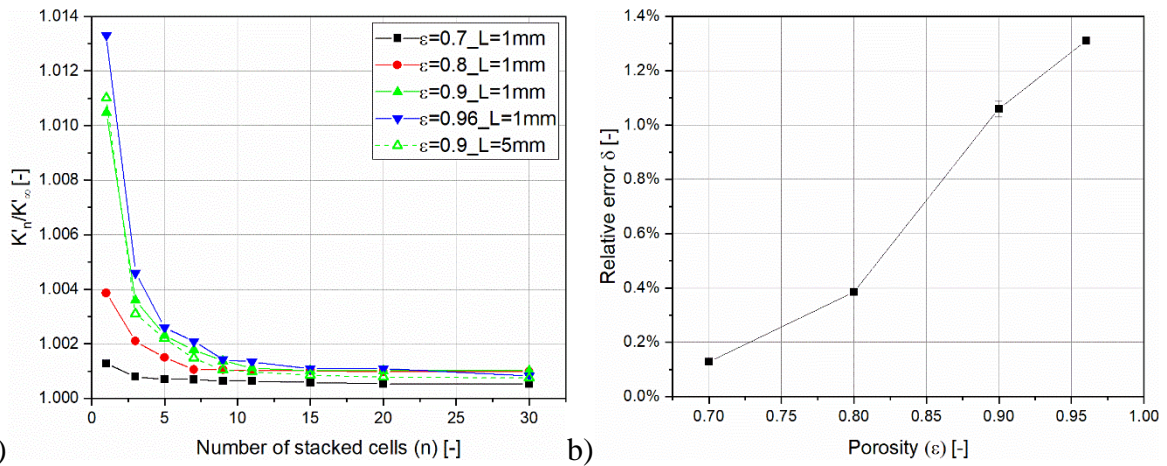


812
 813 Figure 6 Geometrical model for unit cell of PCM composites used for DS simulation and
 814 boundary conditions set to evaluate the permeability K .



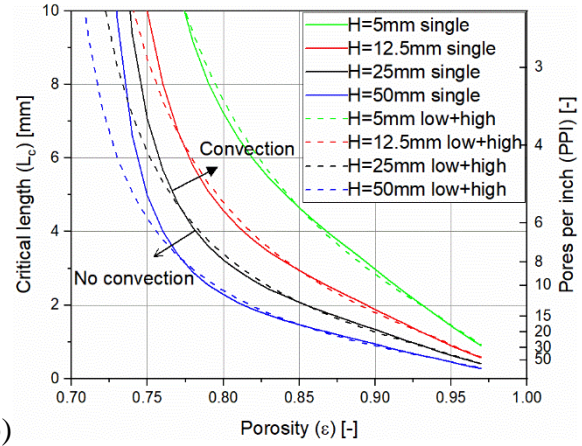
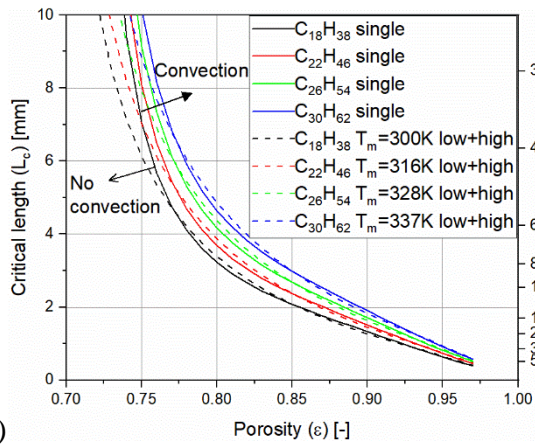
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816 Figure 7. Normalized permeability vs porosity correlation. Datapoints of the present study
 817 given in terms of side length $L=1$ and $L=5\text{mm}$ are compared to experimental data for pore
 818 diameters $75\ \mu\text{m}$ and $400\ \mu\text{m}$ (3:ref[47]), literature DS results (1: ref[12] 2:ref[10]).The best
 819 fit equation 4 refers to Eq.21, 5 is Eq.22, 6 is Eq.23.



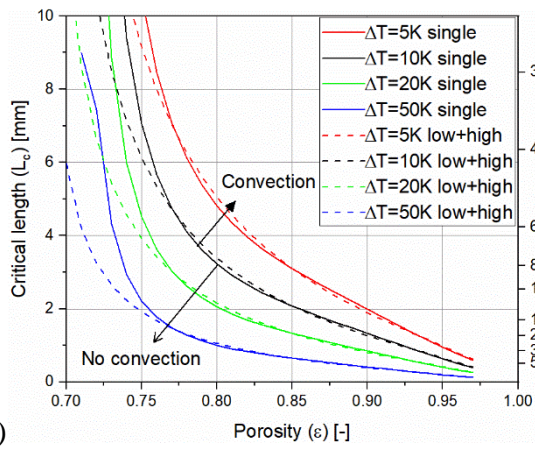
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821 Figure 8 Normalized permeability ratio K'_n / K'_∞ vs. number of stacked cells for BCC structure
 822 for porosity of 0.7 to 0.96 (a) and relative error δ obtained considering K'_∞ instead of K'_1 ,
 823 representing the maximum relative error for porous structures of finite thickness.

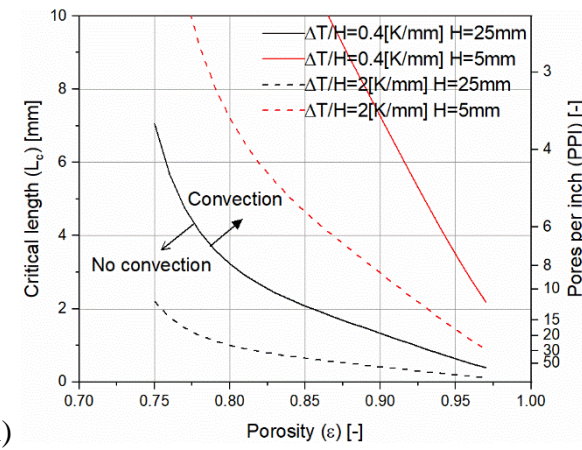


824 a)

b)

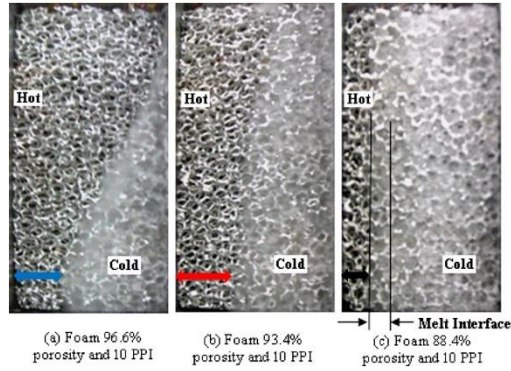


825 c)



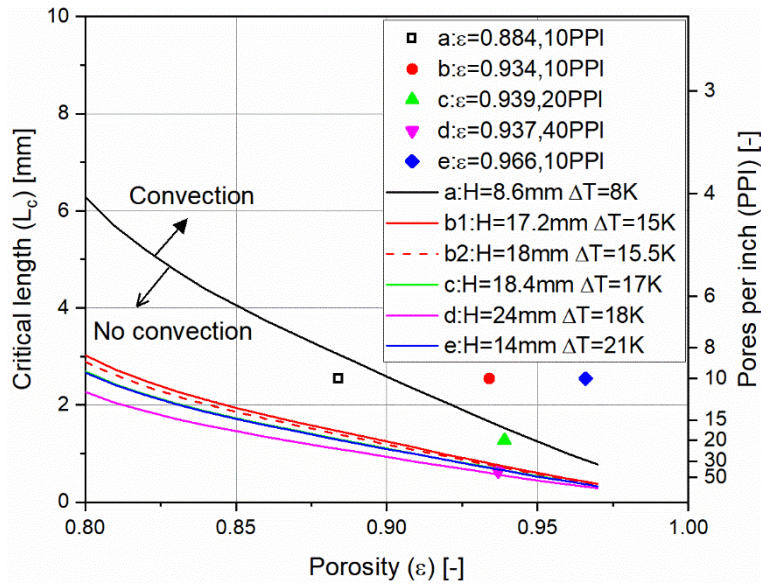
d)

826 Figure 9 Correlations between the critical length L_c and porosity for a reference case (Al/C₁₈H₃₈
 827 paraffin, H=25 mm and $\Delta T=10$ K) in the case of specimen laterally heated are compared to
 828 situations where different paraffins (a), different specimen thickness H (b) or different
 829 temperature difference across the specimen (c) are set. (d) shows L_c - ϵ in terms of the average
 830 temperature gradient $\Delta T/H$ for H=25 mm or 5 mm. In plots a-c lines corresponding to R_D
 831 calculated using K'_∞ for a single (wide) range of porosity are plotted in solid lines (referred as
 832 'single' in the legend), those corresponding to K'_∞ calculated separately for low-and high-
 833 porosity range are plotted as dashed lines (referred as 'low+high' in the legend). Lines in (d)
 834 only refer to single range description of K'_∞ .



835

836 Figure 10 Set 1 PCM composites experimentally tested by Lafdi et al. (reprinted from [25],
 837 with the permission of AIP Publishing) by heating them on vertical surface, shown on the left-
 838 side in these images. For each specimen, the distance arrow is added to show how values of H
 839 are derived.



840

841 Figure 11 Correlations between porosity ϵ and the critical length L_c (or corresponding pore size
 842 PPI) above which natural convection occurs for the experimental tests carried out by Lafdi at
 843 al. [25], compare to points corresponding to samples with the specific porosity and PPI. The
 844 sample with open symbol, laying below the corresponding line, displays no convection.

845