

## Flexible object-oriented modelling for the control of large gas networks

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**Abstract:** In this work we develop an innovative framework for the dynamic modelling of natural gas transmission networks. Thanks to its flexibility, this tool may be used to support operation of the network while also giving the possibility to explore and test new optimized scenarios, introduce new elements in the grid itself (ranging from new branches to new machines) and test the use of new gas mixtures. We use as benchmark the Italian high pressure gas transportation network, executing simulations of past daily operation scenarios. Results are compared to real measurements so to prove the validity of the adopted approach.

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*Keywords:* Natural gas networks; dynamic modelling; simulation; Modelica.

### 1. INTRODUCTION AND MOTIVATION

The energy sector is facing a revolution in the upcoming years. As, at the moment, renewable sources cannot fulfill by their own the whole global energy demand, fighting climate changes requires the introduction of new technologies and an efficiency enhancement for the already present production systems. The major role that natural gas will play in this transition period is largely recognized thanks to its relevant lower heating value and reduced pollutants emissions compared to other fossil fuels (see IEA (2019)). The most common way to transport natural gas is through high pressure pipelines, thanks to gas-powered compression stations that move the fuel from the extraction spots to the delivery points along the grids. Increasing the efficiency of the gas transport system is thus consistent with the goal of progressive reduction of climate-altering gas emissions. This can be obtained for example through daily operation optimization of the integrated asset, by progressively substituting the gas fired compressors with electrically-operated ones and possibly by injecting in the grid larger and larger quantities of hydrogen, allowing to decarbonize also the end-users.

In this paper we propose a comprehensive framework for dynamic modelling and simulation of gas networks, characterized by high flexibility, reusability and extensibility, developed by exclusively using open-source tools.

Such a framework allows to automatically generate models of existing networks with different degrees of detail and accuracy and for different applications (e.g. dynamic simulation, optimal control with direct methods), start-

ing from the network operator's databases, thus enabling improvements in the daily operation and control of the system. It also allows to explore future evolutions of the network in different directions: expansion of the network, addition of new types of machinery (e.g. electrically operated compressors), use of new gas mixtures (e.g. natural gas and hydrogen, or possibly hydrogen alone), with a minimal development effort, thanks to the adopted high-level declarative approach.

The goal of this framework is to eventually support all model-related activities of the network operator in a unified fashion, in particular the daily operation of the current system, as well as studies of its evolution in the near and far future.

The framework was developed with open-source languages and tools, namely Python and Modelica (Mattsson et al., 1998). The first one is a high-level object-oriented programming language that was used to create suitably simplified graph-based representations of the system. The second one is a declarative, object-oriented, multi-domain modelling language used to write the models of the components that are assembled together following the indications stored in the graph, so as to create complete dynamic system models. OpenModelica was the environment chosen to run the simulations (Fritzson et al., 2020).

The case study presented in this paper is the Italian natural gas high pressure network, see Fig. 1. This is a very large, complex and strongly integrated network, managed by a single public operator, with a total length of approx. 8000 km, featuring about 1800 pipes and 6000 nodes



Once the algorithm produces the new graph (shown in figure 2), the corresponding Modelica dynamic model is obtained by instantiating the appropriate physical models corresponding to the surviving edges in the graph, parameterized with the available information, and then connecting them in the same fashion. The physical models of all components are stored in the *GasNetworks* library, which was developed specifically for this work. The final grid model in the studied case accounts for 211 edges and 158 nodes.

**Natural gas model** As the operative pressure range is  $45 \div 75 \text{ bar}$ , real gas effects must be considered by using an equation of state in which the compressibility factor  $Z$  is considered, which normally depends on temperature ( $T$ ), pressure ( $p$ ) and composition. Many authors use an important simplification hypothesis, see Liu et al. (2020), Osiadacz and Chaczykowski (2001): temperature is considered constant along the grid as pipelines are most of the time underground and fixed to  $15^\circ\text{C}$ . This reduces the equation of state to

$$pv = Z(p, \mathbf{x})R^*(\mathbf{x})\bar{T}, \quad (1)$$

where  $R^*$  is the individual gas constant of the natural gas and  $v$  the specific volume. Italy mainly gets natural gas from Russia and Africa, that extract gas with significantly different compositions. Explicit modelling of the transport of individual chemical species is possible, but would increase the computational load substantially. For the purposes of this work, for a given flow configuration, it was possible to approximate  $Z$  (as computed by Lemmon et al. (2018)) as two linear functions of pressure, one for the northern part of the grid and one for the southern one.

**Pipe model** Assuming that natural gas behaves as a compressible, Newtonian, single phase fluid flowing inside infinitely rigid cylindrical pipes and considering the flow completely turbulent, one-dimensional and isothermal, the equations that describe the dynamic phenomena in the grid are the reported mass balance (2), momentum balance (3) and energy balance (4)

$$\frac{dm}{dt} = \int_{\Omega_f} \frac{\partial \rho}{\partial t} d\Omega + \int_{\Omega_f} \nabla \cdot (\rho \mathbf{u}) d\Omega = 0 \quad (2)$$

$$\begin{aligned} \frac{d(m\mathbf{u})}{dt} &= \int_{\Omega_f} \frac{\partial \rho \mathbf{u}}{\partial t} d\Omega + \int_{\Omega_f} \nabla \cdot (\rho \mathbf{u} \mathbf{u}) d\Omega = \\ &= \int_{\Omega_f} \mathbf{f} d\Omega + \int_{S_f} \boldsymbol{\tau} dS - \int_{\Omega_f} \nabla p d\Omega \end{aligned} \quad (3)$$

$$T(x, t) = \bar{T} = 15^\circ\text{C} \quad (4)$$

where  $m$  is the mass,  $\rho$  the fluid density,  $\mathbf{u}$  the velocity vector,  $\Omega_f$  is the control volume,  $S_f$  the control surface,  $\mathbf{f}$  the volume forces and  $\boldsymbol{\tau}$  the shear stresses.

Considering that normal operation gas speed is on average less than  $10 \text{ m/s}$ , the kinetic term in the momentum balance is negligible as suggested by Liu et al. (2020) and Guandalini et al. (2017). The linearized model can be interpreted, as done by Taherinejad et al. (2017), as an electrical equivalent so that a capacitance, inductance and resistance may be identified when interpreting mass flow rate variations as currents and pressure variations as voltages. The results of Peres et al. (1998) for the electrical

case lead to the following transfer function that links the variation of the inlet mass flow rate (imposed by the compression station that can be seen as a current generator when it regulates the mass flow rate) to the variation of the outlet pressure for a *RIC* pipe:

$$\frac{\Delta p_{out}}{\Delta w_{in}} = \frac{\sqrt{\frac{r+si}{sc}}}{\sinh(\sqrt{(r+si)sc} \cdot L)} \quad (5)$$

where  $r$ ,  $c$  and  $i$  are the specific resistance, capacitance and inductance per unit length, respectively, and  $L$  is the length of the considered pipe. To reduce the computational effort, it is possible to neglect the inertial effects, i.e., the left-hand side of Eq. (3), reducing the *RIC* model to an *RC* one. This is valid when the condition presented in equation (6) is satisfied in equation (5).

$$\sqrt{r + j\omega i} \approx \sqrt{r} \rightarrow \omega \ll \frac{1}{\tau_{ri}} \quad (6)$$

where  $\tau_{ri} = i/r$ . Typical values for  $\tau_{ri}$  in the considered case study are below one minute, which is a very fast time constant considering the overall system dynamics, that has time scales of hours, and even the sampling interval of four minutes.

The balance equations resulting from the simplifications are (7) and (8)

$$AL \frac{dp}{dt} = w_{in} - w_{out} - w_{cons} \quad (7)$$

$$p_{out} - p_{in} = c_f \bar{\rho} \omega \frac{u^2 L}{2A} - \bar{\rho} g(z_{out} - z_{in}) \quad (8)$$

where  $A$  is the cross section of the pipe,  $w$  are mass flows,  $z$  the height of inlet and outlet of the pipe,  $c_f$  is the Fanning friction factor,  $\omega$  the wet perimeter and  $w_{cons}$  is the summation of all the consumptions/injections resulting from the simplification algorithm presented in subsection 2.0.1. Dividing the volume representing the pipe into smaller ones allows to be more accurate while assigning the geographical position of the exogenous fluxes and to obtain better pressure profiles describing the ongoing dynamics. This leads to a model built with several *RC* units in series; in particular, we chose the *CRC* representation splitting the capacitance into two halves connected to the boundaries of the resistance placed in the middle.

To determine the appropriate number of finite volumes, a simple system comprising a pipe between two compression stations prescribing the mass flow rate was analyzed. The analytical solution of the linearized partial differential equation describing this system (Polyanin (2001)) is:

$$w(x, t) = \frac{2an\pi}{L^2} \sum_{n=1}^{\infty} \sin\left(\frac{n\pi\xi}{L}\right) \left(1 - \exp\left(-\frac{an^2\pi^2t}{L^2}\right)\right) \quad (9)$$

where time constants can be identified in the exponential time-dependent term: they are function of the parameter  $n$  that describes the spatial modes of the system. Considering the longest pipe in the simplified Italian network model, the minimum number of volumes so that their time constant is way lower than the sampling time for the inputs of the model (four minutes) turned out to be four.

**Compression station model** Each compression station features several radial compressors working in parallel (some of which may be of the same type), a bypass valve, and a network of on/off valves to connect the compressors to pipelines with a certain direction of flow, according to the given flow configuration. Each compressor is driven by a gas turbine, using the flowing gas as fuel. Once the graph simplification algorithm has run, the station can be described by the object diagram of Fig. 3. Physical connections are a-causal, i.e. they describe the flow rate of fluid at a certain pressure (light blue connections) and the exchange of mechanical power at a certain rotational speed (orange connections) without determining *a priori* which are the inputs and outputs of each component model; the Modelica tool figures out automatically how to solve the equations of the connected system.

The compressors are described by maps supplied by the network operator, relating the volumetric flow, the rotational speed, the isentropic head, and the isentropic efficiency, and are represented by suitable polynomials. The gas turbines are also described by polynomials, relating the fuel flow, the mechanical power output and the rotational speed, which is an input provided by the control system. The object-oriented models of the compressor model and of the gas turbine model are written independently, and are then bound to operate at the same speed by the mechanical connection in the object diagram.

Each machine model has an integer input  $M_p$  (orange arrow in the diagram) indicating the number of similar machines operating in parallel, which is easily modelled by multiplying the flow rates and powers by  $M_p$ . In case  $M_p = 0$  the compressor flow rate and the gas turbine power are set to zero. The parallel operation of machines of different type is instead modelled by the explicit parallel connection of the machine models, as shown in Fig. 3.

The station model is completed by a functional description of the control logic, see Fig. 4. The controller receives the  $M_p$  values for each type of machines and three setpoints for the station total flow rate, inlet, and outlet pressure, and computes a master speed signal. Three corresponding PI controllers are implemented with three lead-lag filters, a minimum selector logic and a single integral action block, such that the controller requiring the lower rotational speed wins. The gain of the controller is scheduled on the values of  $M_p$ . The master speed signal output is then dispatched to the active machines, so they all operate at the maximum possible distance from their surge limit. When all  $M_p = 0$ , the bypass valve is operated with a check-valve logic, allowing free flow through the station when the upstream pressure is larger than the downstream one, while preventing back-flows.

Note that introducing electrically operated compressors in such a model is straightforward: one just needs to replace the gas turbine model with a suitable electrical motor model, connecting it exactly in the same way to the compressor mechanical connector.

**Control valve model** Control valves are used in the network to protect downstream lines. They work following either one or two setpoints: prescribed outlet pressure and maximum flow rate or only outlet pressure. In the first

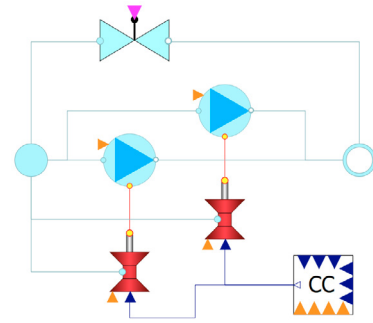


Fig. 3. Object diagram of the compression station model

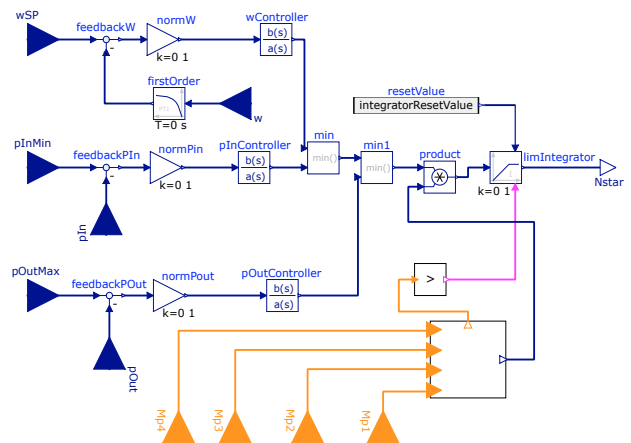


Fig. 4. Block diagram of the compression station controller case, a minimum selector logic similar to Fig. 4 is used, while a standard PI controller is used in the second case.

In order to minimize the model development effort, an idealized linear valve model was used, which allows reproducing the closed-loop behaviour with satisfactory accuracy, while simplifying the task of tuning the controllers and minimizing the need of detailed process data.

### 3. VALIDATION RESULTS

The modelling framework was successfully validated on two portions of the network; two different operative scenarios were tested.

#### 3.1 Long-term scenario

The first validation test is conducted on the southern portion of the grid, see Fig. 5, with an extent of about  $600 \times 500$  km, using recorded data from the system SCADA over a time span of 6 hours.

In this scenario, flows are prescribed to be equal to measurements at every entry point (Mazara, Gela, Melenugno, Fiume Treste) and at the node where the grid is cut (Gallese). Three compression stations are running: Enna runs at maximum rotational speed (because of very high flow set point), while Tarsia and Gallese stations are following the inlet pressure set point. The operating points change slowly over time, due to the dynamically evolving imbalance between the incoming gas flows at the entry points, and the distributed consumptions along the



Fig. 5. Southern portion of the network for validation

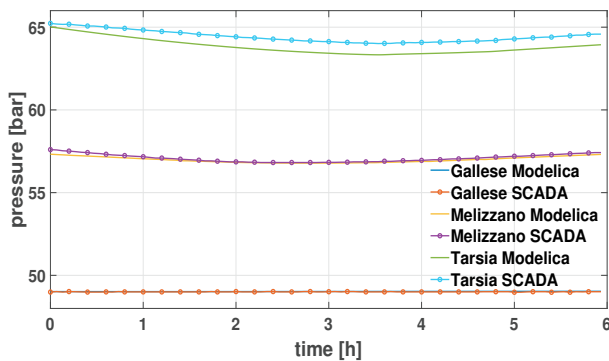


Fig. 6. Pressures along the grid: simulation vs SCADA

network. Selected results are reported to show how the simulated results match the data recorded on the SCADA.

Fig. 6 shows pressure trends at nodes further from the Enna station. Both the values and trends of pressures are reproduced accurately, with relative errors below 1.5%, which are probably due to the effects on gas density (and thus friction losses) of the uncertainty on actual gas composition and temperature and to possible uncertainties of the measurement process.

Fig. 7 and 8 report the simulated and measured rotational speed and fuel flow rate of the Gallese station, which operates under inlet pressure control. Although there are some discrepancies between the two, given the uncertainty on the maps that describe the machines' behaviour, the simulation captures the trends in a satisfactory way.

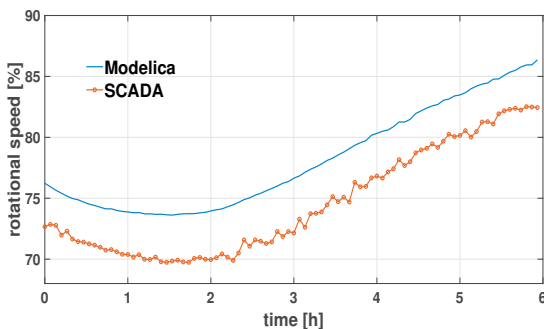


Fig. 7. Rotational speed at Gallese station

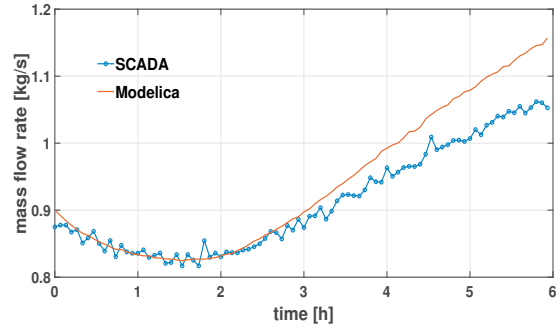


Fig. 8. Fuel consumption of the Gallese station

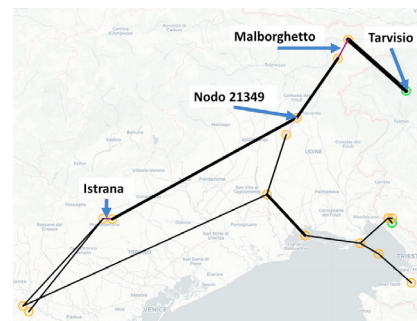


Fig. 9. North-east portion of the network for validation

3.2 Short-term scenario

The goal of the second validation scenario is to confirm that the modelling framework is able to describe faster transients correctly, including complex maneuvers such as turning on a compression station. To this purpose, the section of the grid shown in Fig. 9 was considered. The measured pressures at the boundaries were nearly constant, so the model assumes them as fixed.

In this scenario, the Malborghetto compression station is initially switched off and bypassed, with about 440 kg/s of gas flowing through the bypass. At time  $t = 52 \text{ min}$  one compressor is activated, and the compressor flow rate set point is gradually increased. However, the effect of this increase is not visible outside the station until the flow rate set point overtakes the 440 kg/s at time  $t = 63 \text{ min}$ , because until that moment, the compressor flow just reduces the flow through the bypass. After  $t = 63 \text{ min}$ , the bypass closes and the station starts effectively pumping an increased flow rate into the downstream pipelines.

Fig. 10 shows how the model of the compression station (including the controller) is able to handle this non-trivial transient, using the control logic described in Section 2.0.4. The red curve (bypass flow) and the blue curve (station flow) are initially superimposed. The compressor is then turned on and its flow (yellow curve) is gradually increased, taking over the bypass flow, until eventually the station flow is increased.

Fig. 11 reports the comparison between simulated and recorded pressures: the two top curves represent the pressure at the station outlet, which start increasing after  $t = 63 \text{ min}$ ; the two middle curves represent pressures at node 21348, which is located 30 km downstream the line, and thus react with a certain delay, while the two bottom

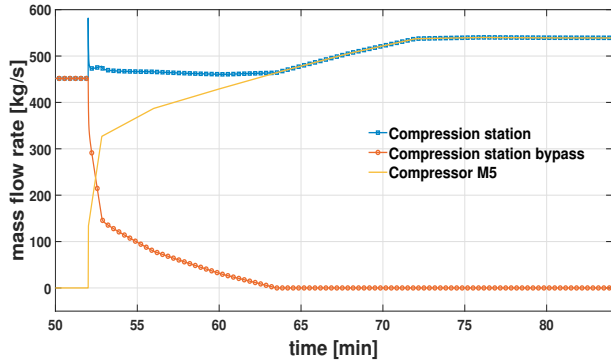


Fig. 10. Mass flow rates in Malborghetto station.

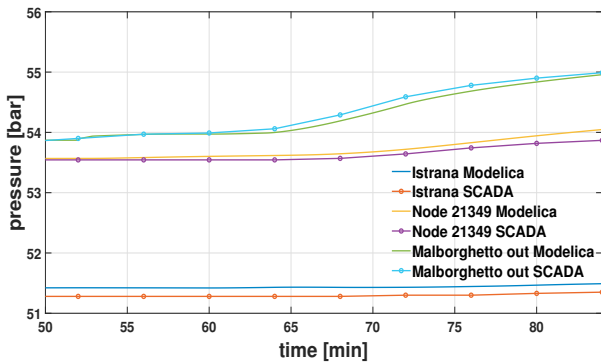


Fig. 11. Pressures in the neighborhood of Malborghetto.

lines represent the pressures at the Istrana station, which are not affected by the transient because the pressure increase has not had yet the time to propagate there within the simulation time span.

#### 4. CONCLUSION

In this paper, a comprehensive dynamic modelling framework for gas networks based on open-source languages and tools has been presented.

The framework is capable to manage complex systems with thousands of components, and to produce suitably simplified equation-based models for simulation purposes, but also to carry out other model-based activities such as direct optimal control. It can then be employed as the foundation for all control, optimization, and planning activities of gas network operators, helping to manage the on-going energy transition.

The tool was validated on the Italian gas network, against measured data obtained from the system SCADA.

The use of the high-level, equation-based, object-oriented Modelica language for the formulation of the physical component model makes the tool flexible and amenable to extensions, such as introducing electrically-operated compressors or natural-gas-hydrogen mixtures, with a limited development effort.

Future work will extend the validation to the entire network; it also may include some of previously reported extensions to the models developed so far, as well as the use of the framework to generate models to solve

direct optimal control problems. The tool could then be integrated with the network operator SCADA, to support the personnel responsible for the daily operation of the system. It could also be employed to support the planning of future extensions of the network and of its use.

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#### REFERENCES

- Fritzson et al. (2020). The OpenModelica integrated environment for modeling, simulation, and model-based development. *Modeling, Identification and Control*, 41(4), 241–295.
- Guandalini, G., Colbertaldo, P., and Campanari, S. (2017). Dynamic modeling of natural gas quality within transport pipelines in presence of hydrogen injections. *Applied Energy*, 185, 1712–1723.
- Hagberg, A., Swart, P., and S Chult, D. (2008). Exploring network structure, dynamics, and function using NetworkX. Technical report, Los Alamos National Lab.(LANL), Los Alamos, NM (United States).
- Herrán-González, A., De La Cruz, J., De Andrés-Toro, B., and Risco-Martín, J. (2009). Modeling and simulation of a gas distribution pipeline network. *Applied Mathematical Modelling*, 33(3), 1584–1600. doi: 10.1016/j.apm.2008.02.012.
- IEA (2019). The role of gas in today’s energy transitions.
- Lemmon, E.W. et al. (2018). NIST Standard Reference Database 23: Reference Fluid Thermodynamic and Transport Properties-REFPROP, Version 10.0, National Institute of Standards and Technology. doi: 10.18434/T4/1502528.
- Liu, K., Biegler, L.T., Zhang, B., and Chen, Q. (2020). Dynamic optimization of natural gas pipeline networks with demand and composition uncertainty. *Chemical Engineering Science*, 215, 115449.
- Mattsson, S.E., Elmquist, H., and Otter, M. (1998). Physical system modeling with Modelica. *Control Engineering Practice*, 6(4), 501–510.
- Osiadacz, A.J. and Chaczykowski, M. (2001). Comparison of isothermal and non-isothermal pipeline gas flow models. *Chemical Engineering Journal*, 81(1-3), 41–51.
- Pambour, K.A., Bolado-Lavin, R., and Dijkema, G.P. (2016). An integrated transient model for simulating the operation of natural gas transport systems. *Journal of Natural Gas Science and Engineering*, 28, 672–690.
- Peres, P.L., Bonatti, I.S., and Lopes, A. (1998). Transmission line modeling: A circuit theory approach. *SIAM review*, 40(2), 347–352.
- Polyanin, A.D. (2001). *Handbook of linear partial differential equations for engineers and scientists*. Chapman and hall/crc.
- Taherinejad, M., Hosseinalipour, S.M., and Madoliat, R. (2017). Dynamic simulation of gas pipeline networks with electrical analogy. *Journal of the Brazilian Society of Mechanical Sciences and Engineering*, 39(11), 4431–4441.