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Evaporation/Condensation Boundary Conditions for the Regularized 13 Moment Equations

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Abstract. The regularized 13 moment equations (R13) are a macroscopic model for the description of rarefied gas flows in the transition regime. The equations have been shown to give meaningful results for Knudsen numbers up to about 0.5. Here, their range of applicability is extended by boundary conditions for evaporating and condensing interfaces, derived from the microscopic interface conditions of kinetic theory. Simple 1-D problems are used to test the R13 equations with evaporation and condensation.

Introduction

The regularized 13 moment (R13) equations are a macroscopic model to describe rarefied gas flows for not too large Knudsen numbers in good approximation to the Boltzmann equation [1, 2, 3, 4, 5]. Gas rarefaction leads to the occurrence of phenomena such as velocity slip and temperature jump at boundaries, Knudsen layers in front of boundaries, transpiration flow, thermal stresses, or heat transfer without temperature gradients, all of which are reproduced by solutions of the R13 equations, but cannot be accurately described by the Navier-Stokes-Fourier (NSF) equations of classical hydrodynamics [3]. Proper modelling of boundary conditions is essential to obtain a meaningful description of rarefied flows, and below we present and test conditions for liquid-vapor boundaries with condensation and evaporation. With this, the range of application of the R13 equations is extended in particular towards microdevices with phase change.

The R13 equations are derived as approximations of the Boltzmann by means of the order of magnitude method [6, 7], which combines elements of the Chapman-Enskog [8, 9, 10, 11, 12] and Grad [13] methods; the resulting equations avoid the problems exhibited by the individual methods.

Just as the transport equations are derived from the Boltzmann equation, the corresponding macroscopic boundary and interface conditions are derived from the microscopic boundary and interface conditions for the Boltzmann equation [14]. For this we use an extended Maxwell boundary model [15] with an condensation/evaporation coefficient and an accommodation coefficient [16]. The derivation follows the same line as that of the wall boundary conditions for non-condensing interfaces. Naturally, for vanishing condensation coefficient, the boundary conditions for solid walls are recovered.

We solve two simple one-dimensional evaporation problems to test the R13 equations with evaporation and condensation. Solutions of R13 and NSF are compared with DSMC simulations, and it is shown that generally R13 gives good agreement to the kinetic solutions, both for overall parameters like the mass and heat flows, and for details of the flows, such as temperature and stress profiles with Knudsen layers.

The R13 equations

The aim of kinetic theory is to find the velocity distribution function $f(x_i, t, c_i)$, where t denotes time t , x_i is the location in space and c_i is the microscopic velocity. In a microscopic approach, the distribution function is the solution of the Boltzmann equation [8, 9], while in a macroscopic approach one derives transport equations as a set of suitable

moments of the Boltzmann equation, where the resulting moment equations are closed by means of an ansatz for the distribution [13, 3]. The basic 13 variables are mass density ρ , macroscopic velocity v_i , temperature T , anisotropic stress tensor σ_{ij} (with $\sigma_{ii} = 0$), and heat flux vector q_i , which are moments of the distribution function, as

$$\rho = m \int f d\mathbf{c} \ , \ \rho v_i = m \int c_i f d\mathbf{c} \ , \ \rho u = \frac{m}{2} \int C^2 f d\mathbf{c} \ , \ \sigma_{ij} = m \int C_{\langle i} C_{j \rangle} f d\mathbf{c} \ , \ q_i = \frac{m}{2} \int C^2 C_i f d\mathbf{c} \quad (1)$$

Here, $u = \frac{3}{2}RT = \frac{3}{2}\theta$ is the specific internal energy, $\theta = RT$ is temperature in energy units with the specific gas constant R , and $C_i = c_i - v_i$ is the peculiar velocity. Indices in angular brackets denote the symmetric, trace-free part of a tensor [3].

The corresponding moment equations are the conservation laws for mass, momentum and energy, which can be written as ($\frac{D}{Dt} = \frac{\partial}{\partial t} + v_k \frac{\partial}{\partial x_k}$ is the material time derivative)

$$\frac{D\rho}{Dt} + \rho \frac{\partial v_k}{\partial x_k} = 0 \ , \quad (2)$$

$$\rho \frac{Dv_i}{Dt} + \rho \frac{\partial \theta}{\partial x_i} + \theta \frac{\partial \rho}{\partial x_i} + \frac{\partial \sigma_{ik}}{\partial x_k} = 0 \ , \quad (3)$$

$$\frac{3}{2} \rho \frac{D\theta}{Dt} + \rho \theta \frac{\partial v_k}{\partial x_k} + \frac{\partial q_k}{\partial x_k} + \sigma_{kl} \frac{\partial v_k}{\partial x_l} = 0 \ , \quad (4)$$

and the balance equations for stress and heat flux

$$\frac{D\sigma_{ij}}{Dt} + \sigma_{ij} \frac{\partial v_k}{\partial x_k} + 2\sigma_{k\langle i} \frac{\partial v_{j \rangle}}{\partial x_k} + \frac{4}{5} \frac{\partial q_{\langle i}}{\partial x_{j \rangle}} + \frac{\partial m_{ijk}}{\partial x_k} = -\rho \theta \left[\frac{\sigma_{ij}}{\mu} + 2 \frac{\partial v_{\langle i}}{\partial x_{j \rangle}} \right] \ , \quad (5)$$

$$\frac{Dq_i}{Dt} + \frac{5}{2} \sigma_{ik} \frac{\partial \theta}{\partial x_k} - \sigma_{ik} \theta \frac{\partial \ln \rho}{\partial x_k} + \theta \frac{\partial \sigma_{ik}}{\partial x_k} + \frac{14}{5} q_{\langle i} \frac{\partial v_{k \rangle}}{\partial x_k} + \frac{4}{3} q_k \frac{\partial v_k}{\partial x_i} + \frac{1}{2} \frac{\partial R_{ik}}{\partial x_k} + \frac{1}{6} \frac{\partial \Delta}{\partial x_i} + m_{ikl} \frac{\partial v_k}{\partial x_l} - \frac{\sigma_{ik}}{\rho} \frac{\partial \sigma_{kl}}{\partial x_l} = -\frac{5}{2} \rho \theta \left[\frac{q_i}{\kappa} + \frac{\partial \theta}{\partial x_i} \right] \quad (6)$$

The collision terms of the above equations were determined for Maxwell molecules, μ is the shear viscosity, and $\kappa = \frac{15}{4}\mu$ is the heat conductivity [3].

For small Knudsen numbers, the Chapman-Enskog method [8, 3] can be used to reduce the equations for stress and heat flux to the Navier-Stokes and Fourier laws,

$$\sigma_{ij} = -2\mu \frac{\partial v_{\langle i}}{\partial x_{j \rangle}} \ , \ \ q_i = -\kappa \frac{\partial \theta}{\partial x_i} \ . \quad (7)$$

In addition to the 13 variables introduced above, the 13 moment equations (3,5,6) contain the additional moments

$$m_{ijk} = m \int C_{\langle i} C_j C_{k \rangle} f d\mathbf{c} \ , \ \Delta = m \int C^4 f d\mathbf{c} - 15\rho\theta^2 \ , \ R_{ij} = m \int C^2 C_{\langle i} C_{j \rangle} f d\mathbf{c} - 7\sigma_{ij} \quad (8)$$

In order to close the system of moment equations, constitutive equations for these variables must be provided. The classical Grad closure [13, 3] simply leads to $m_{ijk|G} = \Delta|_G = R_{ij|G} = 0$. The regularized 13 moment equations arise from an alternative closure, which accounts for parts of, but not the complete, transport equations for $\{m_{ijk}, \Delta, R_{ij}\}$ [1, 6, 7, 17]. The regularized 13 moment equations arise as the appropriate set of equations at 3rd order in the Knudsen number (super-Burnett order), and, for Maxwell molecules, consist of the equations (3,5,6) and the constitutive equations (pressure obeys the ideal gas law, $p = \rho\theta$)

$$\Delta = 5 \frac{\sigma_{kl} \sigma_{kl}}{\rho} + \frac{56}{5} \frac{q_k q_k}{p} - 12\mu\theta \frac{\partial}{\partial x_k} \left(\frac{q_k}{p} \right) \ , \quad (9)$$

$$R_{ij} = \frac{20}{7} \frac{\sigma_{k\langle i} \sigma_{j \rangle k}}{\rho} + \frac{64}{25} \frac{q_{\langle i} q_{j \rangle}}{p} - \frac{24}{5} \mu\theta \frac{\partial}{\partial x_{\langle i}} \left(\frac{q_{j \rangle}}{p} \right) \ , \quad (10)$$

$$m_{ijk} = \frac{4}{3} \frac{\sigma_{\langle ij} q_{k \rangle}}{p} - 2\mu\theta \frac{\partial}{\partial x_{\langle i}} \left(\frac{\sigma_{j \rangle k}}{p} \right) \ . \quad (11)$$

Microscopic and macroscopic boundary conditions

At an evaporating liquid interface, some vapor particles that hit the interface condense, while those that do not condense are reflected; moreover, some particles are injected into the vapor by evaporation from the liquid. We write the distribution function directly in front of the interface as

$$f_{int} = \begin{cases} f^-, & C_n^I \leq 0 \\ f^+, & C_n^I > 0 \end{cases}, \quad (12)$$

where f^- is the distribution of incident particles (negative velocity C_n^I normal, and relative, to the interface), and f^+ is the distribution of emitted particles (positive velocity C_n^I normal to the interface). For finding boundary conditions for moments we shall describe the gas by the distribution function associated with the R13 equations, $f^- = f_{R13}$, which is a Grad distribution for 26 moments, which are the 13 variables plus the 13 constitutive quantities (10) [3].

The distribution of particles leaving the interface is the sum of evaporation and reflection of non-condensing particles back into the vapor. For the latter we follow the classical Maxwell model, which assumes that particles are either specularly reflected, or thermalized and leave in a Maxwellian [15, 16]. The emitted distribution function thus can be written as

$$f^+ = \vartheta f_M(p_{\text{sat}}(\theta_L), \theta_L, C^I) + (1 - \vartheta)(1 - \chi) f_{R13}(C_i^I - 2C_n^I n_i) + (1 - \vartheta)\chi f_M(\bar{p}, \theta_L, C^I), \quad (13)$$

where ϑ is the evaporation/condensation probability, and χ is the accommodation coefficient, defined as the probability that a reflected particle is thermalized. The velocity C_i^I is the velocity of a vapor particle as seen from an observer resting with the liquid-vapor interface; $C_n^I = C_j^I n_j$ is the velocity normal to the interface; $C_i^I - 2C_n^I n_i$ is the specular reflection velocity, the notation is such that $f_{R13}(C_i^I - 2C_n^I n_i)$ denotes the distribution of specularly reflected particles. The pressure \bar{p} in the Maxwellian for thermalized particles is determined from the condition that non-condensing particles must return to the vapor.

The liquid side of the interface is assumed to be in local equilibrium [18, 19], and thus evaporating particles leave in a Maxwellian distribution $f_M(p_{\text{sat}}(\theta_L), \theta_L, C^I)$, where $p_{\text{sat}}(\theta_L)$ is the saturation pressure corresponding to liquid temperature θ_L . For thermal equilibrium, the outgoing distribution reduces to $f_{|E}^+ = f_{R13,E} = f_M(p_{\text{sat}}(\theta_L), \theta_L, C)$.

Continuity conditions for fluxes of moments are used to find the boundary conditions for the moments [13, 14, 4, 5]. For an observer slipping with the gas along the liquid-vapor interface, who observes the particle velocity \hat{C}_i , the normal flux computed with the distribution function directly at the wall, i.e., the distribution f_{int} of Eq. (12), must be equal to the normal flux computed with the distribution function f_{R13} of the gas just in front of the wall [3], which leads to

$$\int_{\hat{C}_n > 0} \hat{\Psi}_A \hat{C}_n f^+ d\mathbf{c} = \int_{\hat{C}_n > 0} \hat{\Psi}_A \hat{C}_n f_{R13} d\mathbf{c}. \quad (14)$$

Here, following Grad [13], we have to use continuity only of fluxes that are odd in \hat{C}_n , which implies functions $\hat{\Psi}_A$ which are even in \hat{C}_n . The 13 variables of the R13 equations are moments based on the weights $\phi_A = m\{1, c_i, c^2, c_i c_j, c^2 c_i\}_A$, and of corresponding the even tensor components are $\phi_{A,\text{even}} = m\{1, c_{i\alpha}, c^2, c_{i\alpha}^2, c_{i\alpha} c_{i\beta}, c^2 c_{i\alpha}\}_A$. We do not discuss any details of the cumbersome derivation of the boundary conditions, but just give the final result, which is obtained after linearization in the evaporation velocity.

As an approximation to the Boltzmann equation, the R13 equations are expected to approximate, but not fully resolve, Knudsen layers [20]. To account for the related inaccuracies, in the interface conditions below we introduce ad-hoc corrections coefficients ϖ_α [21], which are expected to be of order unity. This approach is similar to what is usually done in jump and slip boundary conditions for hydrodynamics [9, 22].

The expression for evaporation flux results for $\hat{\Psi}_A = 1$ as

$$\rho V_n = \varpi_v \frac{\vartheta}{2 - \vartheta} \sqrt{\frac{2}{\pi}} \left(\frac{p_{\text{sat}}(\theta_L)}{\sqrt{\theta_L}} - \frac{\Pi}{\sqrt{\theta}} \right), \quad (15)$$

with the correction coefficient ϖ_v . Hence, the evaporation flux is determined through the difference between the saturation pressure $p_{\text{sat}}(\theta_L)$ of the liquid at the interface and the effective pressure $\Pi = \rho\theta + \frac{1}{2}\sigma_{nn} - \frac{1}{120}\frac{\Delta}{\theta} - \frac{1}{28}\frac{R_{nn}}{\theta}$. This expression is a generalization of the classical Hertz-Kundsen-Schrage law [23, 24, 25, 26] to the R13 equations.

The other conditions are generalizations of the established wall boundary conditions for the R13 equations [14, 5], to which they reduce for non-evaporating interfaces ($\vartheta = 0$):

Generalized slip condition:

$$\sigma_{t_\alpha n} = -\varpi_\sigma \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi\theta}} \left(\Pi V_{t_\alpha} + \frac{1}{5} q_{t_\alpha} + \frac{1}{2} m_{t_\alpha nn} \right) - \rho V_n V_{t_\alpha}, \quad (16)$$

generalized temperature jump condition:

$$q_n = -\varpi_q \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi\theta}} \left(2\Pi(\theta - \theta_L) - \frac{\Pi}{2} V_t^2 + \frac{1}{2} \theta \sigma_{nn} + \frac{\Delta}{15} + \frac{5}{28} R_{nn} \right) + \left[\frac{1}{2} (V_t^2 - \theta_L) - \frac{5}{2} (\theta - \theta_L) \right] \rho V_n, \quad (17)$$

generalized interface conditions for higher moments:

$$m_{nnn} = \varpi_{mn} \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi\theta}} \left[\frac{2}{5} \Pi(\theta - \theta_L) - \frac{3}{5} \Pi V_t^2 - \frac{7}{5} \theta \sigma_{nn} + \frac{\Delta}{75} - \frac{1}{14} R_{nn} \right] - \frac{2}{5} \left[\theta_L + \frac{3}{2} V_t^2 \right] \rho V_n \quad (18)$$

$$m_{t_\alpha t_\beta n} = -\varpi_{mt} \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi\theta}} \left[\theta \sigma_{t_\alpha t_\beta} - \Pi V_{t_\alpha} V_{t_\beta} + \frac{R_{t_\alpha t_\beta}}{14} + \left(\frac{1}{5} \Pi(\theta - \theta_L) + \frac{1}{5} \Pi V_t^2 - \frac{1}{5} \theta \sigma_{nn} + \frac{\Delta}{150} \right) \delta_{\alpha\beta} \right] + \rho V_n \left[V_{t_\alpha} V_{t_\beta} + \frac{1}{5} (\theta_L - V_t^2) \delta_{\alpha\beta} \right] \quad (19)$$

$$R_{t_\alpha n} = \varpi_R \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi\theta}} \left[\Pi \theta V_{t_\alpha} - \frac{11}{5} \theta q_{t_\alpha} - \frac{1}{2} \theta m_{t_\alpha nn} - \Pi V_{t_\alpha} V_{t_\beta} V_{t_\beta} + 6 \Pi V_{t_\alpha} (\theta - \theta_L) \right] + \left[7(\theta - \theta_L) + \theta_L - V_{t_\beta} V_{t_\beta} \right] \rho V_{t_\alpha} V_n \quad (20)$$

The index n refers to the interface normal, and indices t_α with $\alpha = 1, 2$ indicate the two tangential directions.

Interface conditions for hydrodynamics

In the hydrodynamic limit, only first order contributions in the Knudsen number are retained in the equations. For one-dimensional processes, with transport only normal to the interface, and for only small deviations from equilibrium, the interface conditions are often written as [27, 16]

$$\begin{bmatrix} \frac{p_{\text{sat}}(\theta_L) - p}{\sqrt{2\pi\theta_L}} \\ \frac{p}{\sqrt{2\pi\theta_L}} \frac{\theta_L - \theta}{\theta_L} \end{bmatrix} = \begin{bmatrix} \hat{r}_{11} & \hat{r}_{12} \\ \hat{r}_{21} & \hat{r}_{22} \end{bmatrix} \begin{bmatrix} [\rho V_n]_{\text{NSF}} \\ [q_n]_{\text{NSF}} \end{bmatrix} \quad \text{with} \quad \hat{r}_{\alpha\beta|\text{corr}} = \begin{bmatrix} \frac{1}{\vartheta} - 0.40044 & 0.126 \\ 0.126 & 0.291 \end{bmatrix}_{\alpha\beta}, \quad (21)$$

with a symmetric matrix $\hat{r}_{\alpha\beta}$ of Onsager coefficients, or resistivities [28]. The above values for $\hat{r}_{\alpha\beta|\text{corr}}$ are determined from exact calculations based on kinetic theory yield explicit corrections to account for Knudsen layer effects [19, 27], uncorrected values can be found in [16].

1-D heat and mass transfer problems

To put the R13 equations with evaporation/condensation to test, we now consider flows in simple one-dimensional geometry, and steady state. We ignore all details of mass and heat transfer through the liquid, and consider the temperature of the liquid at the interface as given. In particular, we consider systems where the normal of the interface points into the $x_1 = x$ direction, and all flow properties are functions only of this coordinate. Moreover, we consider small deviations from an equilibrium rest state where vapor and liquid are at temperature θ_0 , hence the reference pressure is $p_0 = p_{\text{sat}}(\theta_0) = \rho_0 \theta_0$. We use the rest state data and the length scale L to make the variables dimensionless. With all flows only in x -direction, the variable space is reduced to

$$\hat{\rho}(\hat{x}) = \frac{\rho(L\hat{x})}{\rho_0}, \quad \hat{\theta}(\hat{x}) = \frac{\theta(L\hat{x})}{\theta_0}, \quad \hat{v}(\hat{x}) = \frac{v_x(L\hat{x})}{\sqrt{\theta_0}}, \quad \hat{\sigma}(\hat{x}) = \frac{\sigma_{xx}(L\hat{x})}{p_0}, \quad \hat{q}(\hat{x}) = \frac{q_x(L\hat{x})}{p_0 \sqrt{\theta_0}}. \quad (22)$$

For convenience, the hats indicating dimensionless quantities will be omitted from now on.

The linearized dimensionless conservation laws (3) are easily integrated to give constant mass flow J_0 , constant normal stress P_0 and constant energy flux Q_0 .

$$\rho v = J_0 = \text{const} \quad , \quad p + \sigma = P_0 = \text{const} \quad , \quad \frac{5}{2} J_0 \theta + q = Q_0 = \text{const}. \quad (23)$$

With this, the (linearized, dimensionless, one-dimensional) R13 constitutive equations for Maxwell molecules (10) reduce to

$$\Delta = -12\text{Kn} \frac{\partial q}{\partial x} = 30J_0\text{Kn} \frac{\partial \theta}{\partial x} \simeq 0 \quad , \quad R = -\frac{16}{5}\text{Kn} \frac{\partial q}{\partial x} = 8J_0\text{Kn} \frac{\partial q}{\partial x} \simeq 0 \quad , \quad m = -\frac{4}{5}\text{Kn} \frac{\partial \sigma}{\partial x}. \quad (24)$$

The linearized balance equations for xx -component of stress (5) and x -component of heat flux (6) reduce to

$$\frac{4}{5}\text{Kn} \frac{\partial^2 \sigma}{\partial x^2} = \frac{\sigma}{\text{Kn}} \quad , \quad \frac{\partial \theta}{\partial x} = -\frac{4\hat{Q}_0}{15\text{Kn}} - \frac{2}{5} \frac{\partial \sigma}{\partial x}. \quad (25)$$

where $\hat{Q}_0 = Q_0 - \frac{5}{2}J_0$. These can be integrated easily to give

$$\theta = K - \frac{4\hat{Q}_0 x}{15\text{Kn}} - \frac{2}{5}\sigma \quad , \quad \sigma = A \sinh \left[\sqrt{\frac{5}{4}} \frac{x}{\text{Kn}} \right] + B \cosh \left[\sqrt{\frac{5}{4}} \frac{x}{\text{Kn}} \right] \quad (26)$$

where K, A, B are constants of integration. We note that σ is of Knudsen layer type, i.e., it decays exponentially away from the wall on the scale of the mean free path. We also note that classical hydrodynamics gives $\sigma = 0$, and $\theta = K - \frac{4\hat{Q}_0 x}{15\text{Kn}}$, which is the case for $A = B = 0$.

For the full solution, we have to find the six constants of integration $\{J_0, P_0, \hat{Q}_0, K, A, B\}$, from the appropriate boundary conditions for both sides. Equations (15 - 20) reduce in linearized and dimensionless form to (no tangential components)

$$J_{0,n} = \varpi_v \frac{\vartheta}{2 - \vartheta} \sqrt{\frac{2}{\pi}} \left(p_{\text{sat}}(\theta_L) - P_0 - \frac{1}{2}(\theta_L - \theta) + \frac{1}{2}\sigma \right), \quad (27)$$

$$\hat{Q}_{0,n} = \varpi_q \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi}} \left(2(\theta_L - \theta) - \frac{1}{2}\sigma \right) - \frac{1}{2}J_{0,n}, \quad (28)$$

$$\frac{4}{5}\text{Kn} \left[\frac{\partial \sigma}{\partial x} \right]_n = \varpi_{mn} \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi}} \left[\frac{2}{5}(\theta_L - \theta) + \frac{7}{5}\sigma \right] + \frac{2}{5}J_{0,n}. \quad (29)$$

Here it is assumed that all dimensionless pressures $p_{\text{sat}}(\theta_L)$, P_0 , and all dimensionless temperatures θ_L, θ , are close to unity. $J_{0,n}, \hat{Q}_{0,n}$ are the products of the flows J_0, \hat{Q}_0 with the normal at the respective boundary. For NSF, we consider the interface conditions (21).

Half space problem

In the classical problem of the steady evaporation from a planar infinite surface into a half-space [29, 30, 31], the evaporating liquid surface is kept at temperature θ_L with evaporation pressure $p_{\text{sat}}(\theta_L)$. It is assumed that far downstream the flow is in a uniform equilibrium state characterized by pressure $p_\infty \leq p_{\text{sat}}$, temperature θ_∞ and bulk velocity $v_\infty > 0$. Previous investigations [29] have shown that steady evaporation is possible only when the downstream flow is subsonic and that only one of the three downstream flow parameters can be freely assigned. Therefore, the required solution is obtained by setting $Q_0 = \sigma_\infty = 0$ and hence the solution of the problem follows from (26) as:

$$\theta(x) = \theta_\infty - \frac{2}{5}\sigma(x) \quad , \quad \sigma(x) = A \exp \left[-\sqrt{\frac{5}{4}} \frac{x}{\text{Kn}} \right]. \quad (30)$$

With the evaporation speed v_∞ prescribed, the downstream temperature θ_∞ and the pressure p_∞ are fully determined by the relationships:

$$\alpha_p = \frac{p_{\text{sat}}(\theta_L^0) - p^\infty}{v_\infty / \sqrt{2}} \quad , \quad \alpha_\theta = \frac{\theta_L - \theta_\infty}{v_\infty / \sqrt{2}} . \quad (31)$$

The coefficients α_p and α_θ can be determined by solving the linearized Boltzmann equation. For the Maxwell molecules potential considered here, an accurate approximate solution, for unit evaporation coefficient, has been obtained by Ytrehus by a moment method. Therefore, we compare results from R13 with Maxwell molecules and BGK model [32], Ytrehus' solution (Y) [29], and NSF, and will use the comparison to fit the corrections coefficients ϖ_α . Since we only have two coefficients α_p , α_θ , we have some freedom of choice, and, to keep things simple, we chose $\varpi_{mn} = 1$, and fit only ϖ_V , ϖ_q . The various methods yield for the coefficients

$$\alpha_{p|R13} = \frac{\sqrt{\pi}}{4} \left[\frac{1}{2} \frac{1}{\varpi_q} + \frac{4}{\varpi_V} + \frac{\frac{2}{5} \left(1 + \frac{1}{4\varpi_q}\right)}{1 + \frac{2}{15} \sqrt{10\pi}} \right] \quad , \quad \alpha_{p|Y} = \alpha_{p|NSF} = 2 \sqrt{\pi} \hat{r}_{11} = \frac{2}{\sqrt{\pi}} + \frac{9\sqrt{\pi}}{16} = 2.1254$$

$$\alpha_{\theta|R13} = \frac{\sqrt{\pi}}{4} \left[\frac{1}{\omega_q} + \frac{\frac{4}{25} \left(1 + \frac{1}{4\omega_q}\right)}{1 + \frac{2}{15} \sqrt{10\pi}} \right] \quad , \quad \alpha_{\theta|Y} = \frac{\sqrt{\pi}}{4} = 0.44311 \quad , \quad \alpha_{\theta|NSF} = 2 \sqrt{\pi} \hat{r}_{12} = 0.44723$$

For NSF the coefficients α_p and α_θ are, apart from the factor $2\sqrt{\pi}$, just the dimensionless resistivities \hat{r}_{11} and \hat{r}_{12} as defined in (21). In the R13 equations, we fit the free correction coefficients ϖ_V , ϖ_q , to the Ytrehus values for Maxwell molecules, to find $\varpi_V = 0.9822$, $\varpi_q = 1.126$. As expected, the correction coefficients are of order unity. Without the correction, i.e., for $\varpi_V = \varpi_q = 1$, the R13 equations yield $\alpha_p = 2.1208$, $\alpha_\theta = 0.49383$, which is in good agreement for the pressure coefficient, but gives a 10% deviation for the temperature coefficient.

Heat and mass transfer between two reservoirs

For a closer look at the Knudsen layer, we consider heat and mass transfer between two liquid reservoirs at (dimensionless) locations $x = \pm \frac{1}{2}$, where the quantities at the two interfaces are indicated with superscripts 0,1. The solution requires interface conditions on both sides of the domain. Note that at $x = \frac{1}{2}$ the interface normal points into the negative direction, so that, i.e., $J_{0,n}(x - \frac{1}{2}) = -J_{n,0}(x + \frac{1}{2}) = J_0$ etc. We have three pairs of boundary conditions for V_n , q_n , m_{nn} , given by (28), which are best applied by taking their pairwise sums and differences, respectively. After some calculation, the solution of the linear problem can be presented as

$$P_0 = \frac{p_{\text{sat}}(\theta_L^0) + p_{\text{sat}}(\theta_L^1)}{2} \quad , \quad \theta = \frac{\theta_L^0 + \theta_L^1}{2} - \frac{4\hat{Q}_0}{15\text{Kn}} x - \frac{2}{5} A \sinh \left[\sqrt{\frac{5}{4}} \frac{x}{\text{Kn}} \right] \quad , \quad \sigma = A \sinh \left[\sqrt{\frac{5}{4}} \frac{x}{\text{Kn}} \right] . \quad (32)$$

Two of the pairwise sums of the boundary conditions give $0 = (\theta_L^0 - \theta^0) + (\theta_L^1 - \theta^1)$, which was used in the above.

The evaporation fluxes J_0 , the heat flux Q_0 and the amplitude of the Knudsen layer A are obtained from a linear system which results from subtracting the interface conditions at both sides:

$$J_0 = \varpi_V \frac{\vartheta}{2 - \vartheta} \sqrt{\frac{2}{\pi}} \frac{1}{2} \left(p_{\text{sat}}(\theta_L^0) - p_{\text{sat}}(\theta_L^1) + \frac{2Q_0}{15\text{Kn}} + \frac{\theta_L^1 - \theta_L^0}{2} - \frac{3}{5} A \sinh \left[\sqrt{\frac{5}{4}} \frac{1}{2\text{Kn}} \right] \right) , \quad (33)$$

$$\hat{Q}_0 = -\varpi_q \frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)} \sqrt{\frac{2}{\pi}} \left(\frac{4Q_0}{15\text{Kn}} + \theta_L^1 - \theta_L^0 + \frac{3}{10} A \sinh \left[\sqrt{\frac{5}{4}} \frac{1}{2\text{Kn}} \right] \right) - \frac{J_0}{2} , \quad (34)$$

$$A = -\varpi_{mn} \frac{\frac{\vartheta + \chi(1 - \vartheta)}{2 - \vartheta - \chi(1 - \vartheta)}}{2\sqrt{5} \cosh \left[\sqrt{\frac{5}{4}} \frac{1}{2\text{Kn}} \right]} \sqrt{\frac{2}{\pi}} \left(\frac{4Q_0}{15\text{Kn}} + \theta_L^1 - \theta_L^0 + \frac{39}{5} A \sinh \left[\sqrt{\frac{5}{4}} \frac{1}{2\text{Kn}} \right] \right) + 2J_0 . \quad (35)$$

Solving the system, and inserting the results into (32) yields detailed profiles of temperature and stress. The NSF solution results by replacing (34)₃ with $A = 0$. We compare R13 and NSF predictions to DSMC simulations (all with

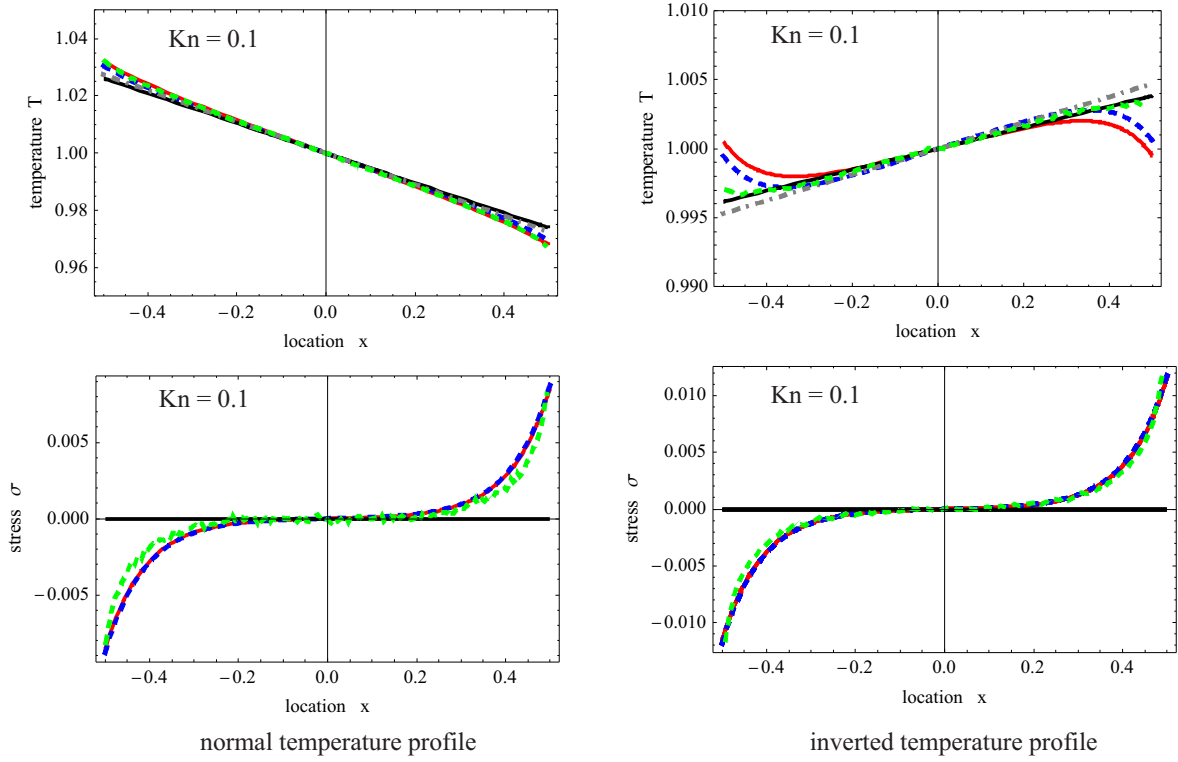


FIGURE 1. Normal (left) and inverted (right) temperature profile: Temperature and stress profiles for $Kn=0.1$. Comparison of DSMC (green, dashed), R13 with corrected BC (red, continuous), R13 with uncorrected BC (blue, dashed), NSF with Knudsen layers correction (black, continuous), NSF without correction (grey).

$\chi = 1$) for cases with normal temperature profile, and inverted temperature profile. Dimensionless temperatures and saturation pressure of the two liquids are prescribed as $\theta_L^{0,1} = 1 \pm \Delta\theta$, $p_{\text{sat}}(\theta_L^{0,1}) = 1 \pm \Delta p$, with $\Delta p = 0.075$, and $\Delta\theta = 0.05$ for the normal temperature profile, $\Delta\theta = 0.1$ for the inverted profile.

Figure 1 shows on the left temperature and stress profiles for the case of normal temperature profile, which has the temperature gradient pointing from hot (left) to cold (right), comparing DSMC with R13 and NSF solutions for $Kn = 0.1$. The figures indicates good agreement of R13 results with DSMC for temperature and stress profiles, with somewhat better agreement for the boundary conditions with adjusted correction coefficients ϖ_α (red curve) over the uncorrected coefficients ($\varpi_\alpha = 1$, blue). NSF with (black) and without (grey) Knudsen layer correction cannot match the temperature curve, and give $\sigma = 0$, while R13 can match the stress quite well. The upper block in Table 1 gives the corresponding values of mass and heat flux, and it becomes clear that all theories give the mass flux J_0 in good agreement to DSMC, while the heat flux \hat{Q}_0 exhibits considerable deviations.

The plots on the right of Figure 1, and the lower block of Table 1 show results for the inverted temperature profile, which has the temperature gradient pointing from cold (right) to hot (left), again comparing DSMC with R13 and NSF solutions for $Kn = 0.1$. In this case, the R13 equations with (red) and without (blue) correction overemphasize the

TABLE 1. Mass and heat flux, and relative error to DSMC in %, for vapor enclosed between two liquid surfaces at $Kn = 0.1$, determined from DSMC, R13, and NSF.

		DSMC	R13, corr	error	R13	error	NSF, corr	error	NSF	error
normal	J_0	0.0294	0.0289	1.7	0.0287	0.7	0.0303	3.1	0.0310	5.4
	\hat{Q}_0	0.0172	0.0215	25	0.0203	18	0.0196	14	0.0206	20
inverted	J_0	0.0503	0.0506	0.6	0.0510	1.4	0.0504	0.2	0.0540	7.4
	\hat{Q}_0	-0.00274	-0.00307	12	-0.00395	44	-0.00288	5.1	-0.00353	29

Knudsen layer for the temperature, while NSF with correction (black) gives an excellent match. While NSF gives $\sigma = 0$, the R13 prediction for stress agrees well with DSMC. All theories give the mass flux J_0 in good agreement to DSMC, while the heat flux \hat{Q}_0 exhibits considerable deviations. Interestingly, NSF with corrected boundary conditions gives the best results, while NSF with uncorrected boundary conditions deviates most. This points to the importance of adjusting the boundary conditions properly.

Conclusions

The evaluation of evaporation/condensation processes with the R13 and NSF equations based on non-equilibrium boundary conditions shows that both can produce results in reasonable agreement with DSMC calculations. R13 can, in principle, resolve more detail, including Knudsen layers for the temperature, and—in particular—for normal stress, which is not accessible for NSF. It should be noted that the NSF boundary conditions used are carefully corrected based on the 2nd law of thermodynamics and fitting to exact solutions, which give the Onsager coefficients in the matrix $\hat{r}_{\alpha\beta}$. A similar fitting procedure will be required for the R13 equations, which here were corrected only in a rather ad-hoc way, by means of the coefficients ϖ_α . A more careful study of the boundary conditions based on a full 2nd law analysis with proper Onsager coefficients [20] should lead to improved agreement between R13 and DSMC, or other exact solutions of the Boltzmann equation.

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